

saipem



Effect of Underwater Explosion on Pipeline Integrity

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OUTLINE

- **UNEXPLODED ORDNANCES**
- **RECENT STUDIES**
- **UNDERWATER EXPLOSION**
- **OBJECTIVE AND SCOPE OF WORK**
- **ABAQUS FEM MODEL DESCRIPTION AND VALIDATION**
- **APPLICATION**
 - **Scenario Description**
 - **FEM Analysis Results**
- **CONCLUSIONS**
- **FUTURE DEVELOPMENTS**



UNEXPLODED ORDNANCES – WHAT IS IT?

Definition of **UneXploded Ordnance (UXO)** is given by United Nations as follows:

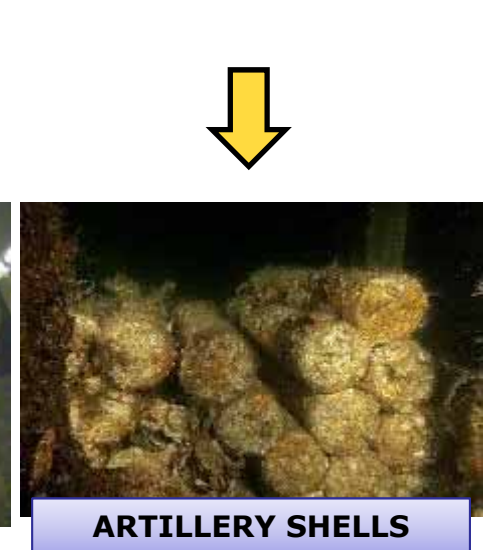
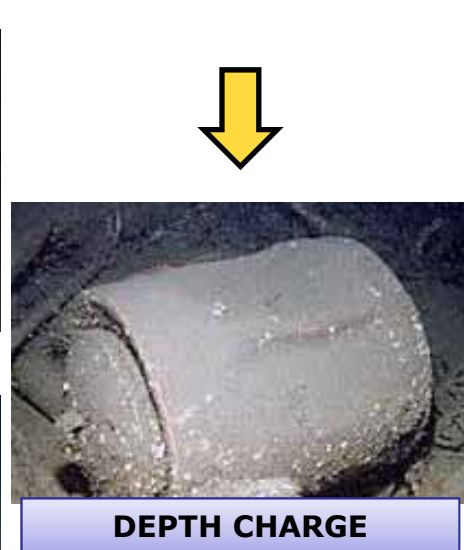
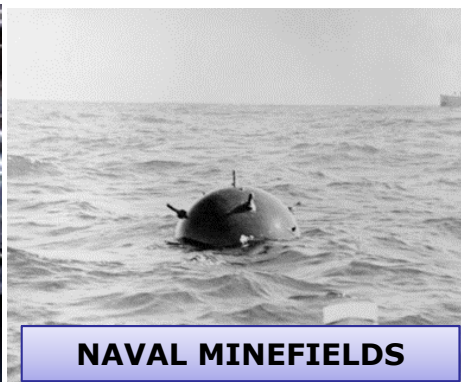
«...explosive ordnance that has been primed, fused, armed, or otherwise prepared for use and used in an armed conflict. It may have been fired, dropped, launched or projected and should have exploded but failed to do so»

Found UXOs originate from three principal sources:

1. **Military training exercises** (abandoned gunnery ranges, naval warfare exercises);
2. **Accidental disposal** due to poor working practices during munitions handling and transportation, or other accidental events (shipwreck, crash landing, ecc.);
3. **Wartime ops** during armed conflicts (WWI and WWII mainly), including:
 - Naval ship **bombing and torpedoing** events;
 - **Anti-submarine** warfare;
 - Long range **shelling** (naval gunnery, coastal artillery);
 - Munitions deliberately placed as means of **area denial** (naval mine fields);
 - Munitions deliberately sunk by warring armies to avoid enemy appropriation.

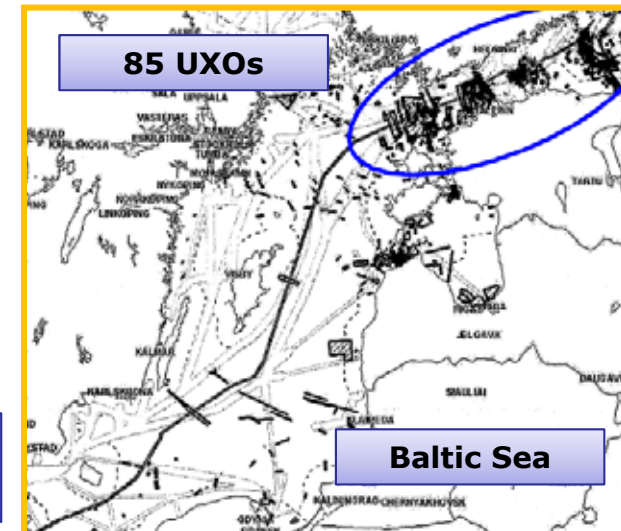
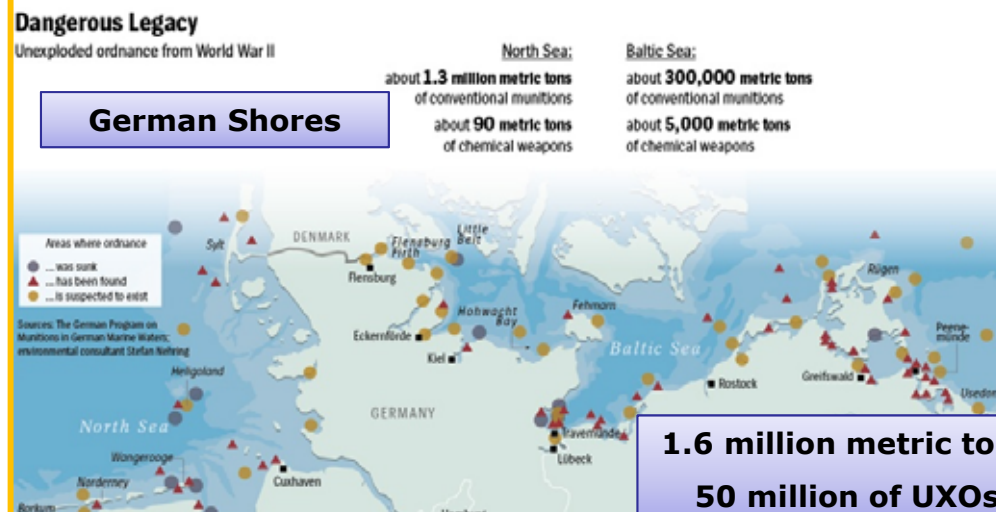


UNEXPLODED ORDNANCES – WARTIME ORIGINS



UNEXPLODED ORDNANCES – WHERE?

UXO arises from both hostile and defensive **MILITARY ACTIVITIES** often related to World Wars I and II. Their occurrence is higher in documented **WAR THEATRE** sea regions (e.g. Baltic Sea, North Sea, shores of Northern Germany, English Channel, Mediterranean Sea, Western Areas of Pacific Ocean, ecc.), or in disused **FIRE RANGES**.



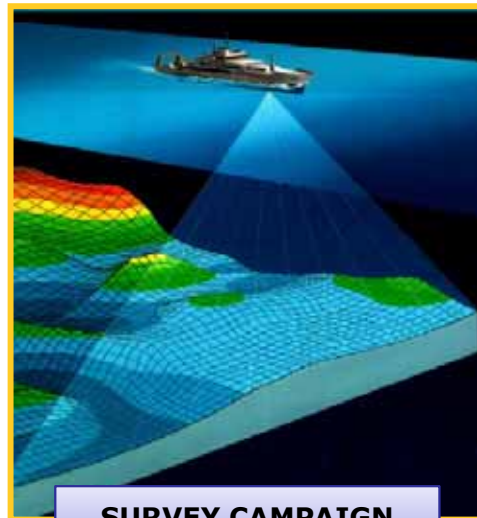
**1.6 million metric tons
50 million of UXOs**



UNEXPLODED ORDNANCES DISCOVERY – SURVEY

During survey campaign activities **UNEXPLODED ORDNANCES (UXOs)** are **FREQUENTLY** discovered.

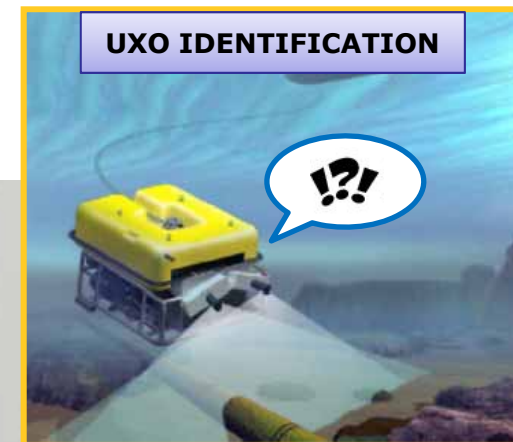
As they are a **HIGH CONSEQUENCE** but **LOW PROBABILITY** event, appropriate allowance should be made for assessing the risk of encountering UXO on-site and for mitigating that risk if significant.



SURVEY CAMPAIGN



SUSPECT OBJECTS



UXO IDENTIFICATION



ASSESSMENT OF RISK OF ACCIDENTAL EXPLOSION AND PIPELINE INTEGRITY



UNEXPLODED ORDNANCES – CHARACTERIZATION

MASS OF EXPLOSIVE CHARGE

Mass Range: 15 – 1000 kg

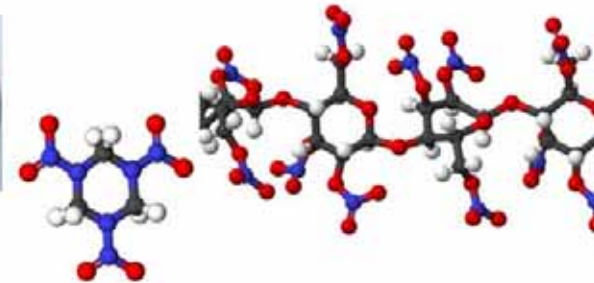
Mass Average: 200 – 300 kg



WARHEAD EXPLOSIVE TYPE

TNT, Hexanite, Nitrocellulose, RDX, Torpex

(Often it is difficult to determine the correct explosive type)

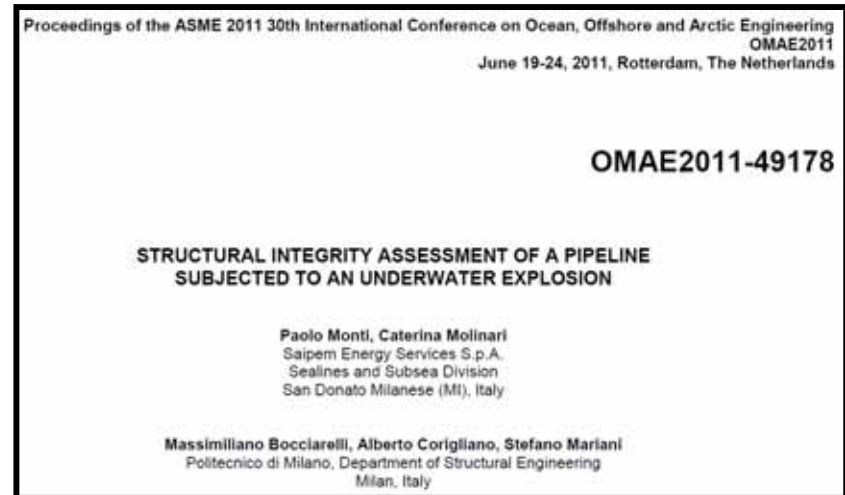
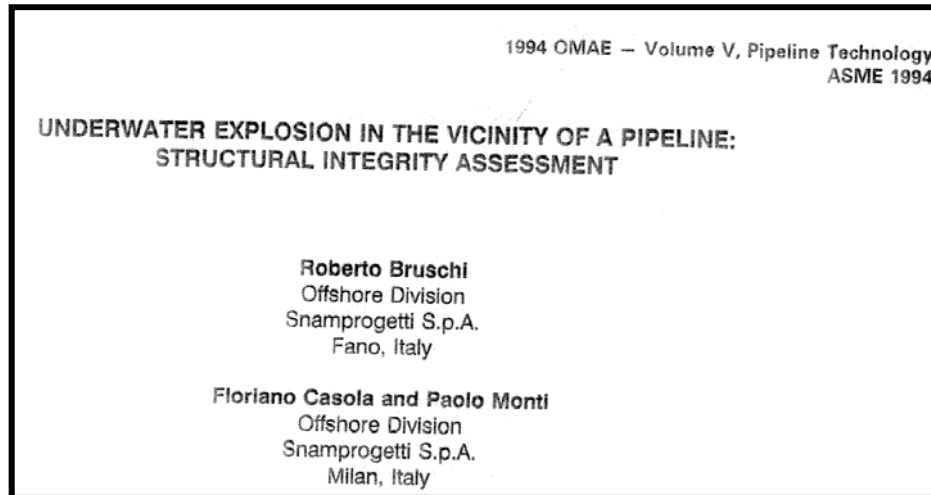


WARHEAD SHAPE AND DIMENSIONS

High variability depending from the UXO type



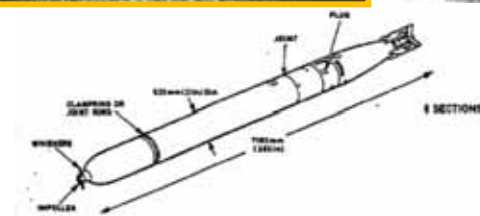
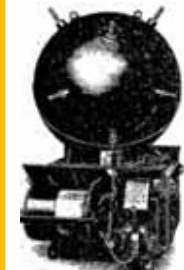
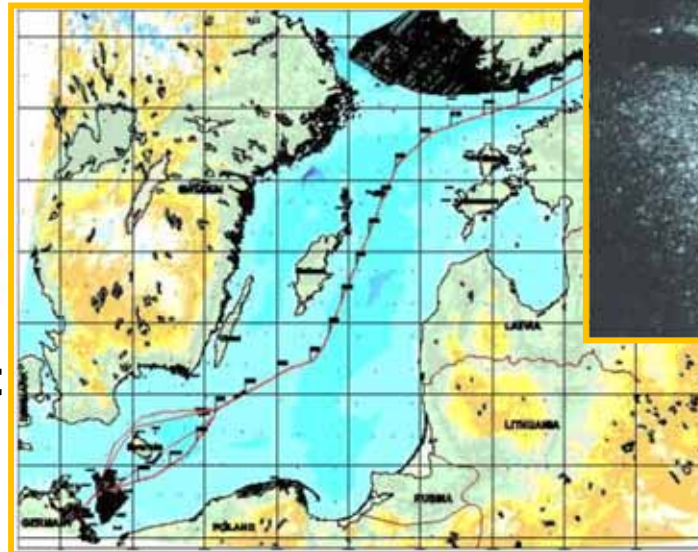
RECENT STUDIES



TransMed

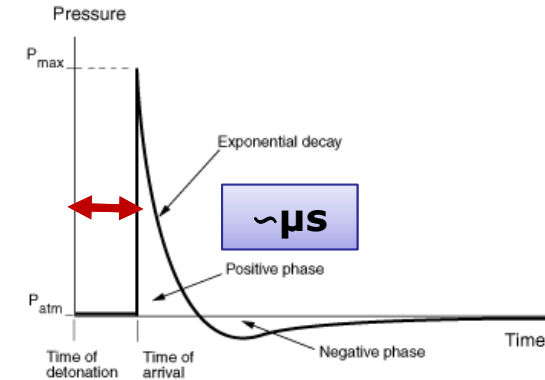
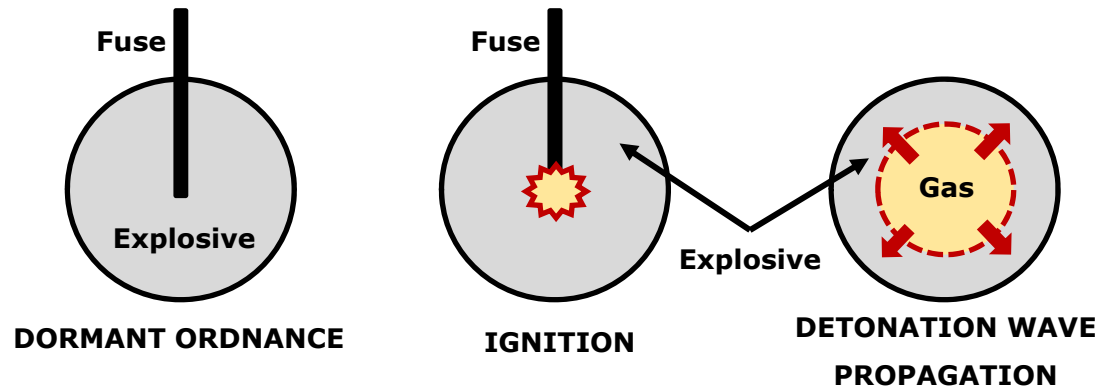


Nord Stream Project

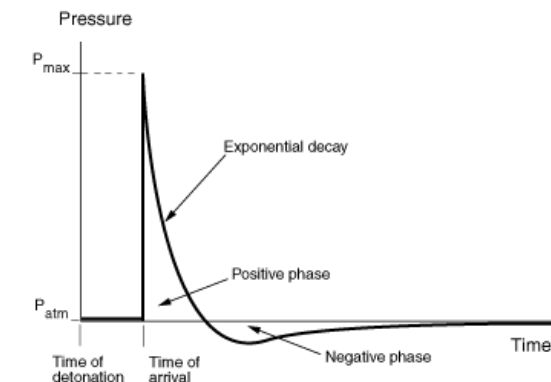
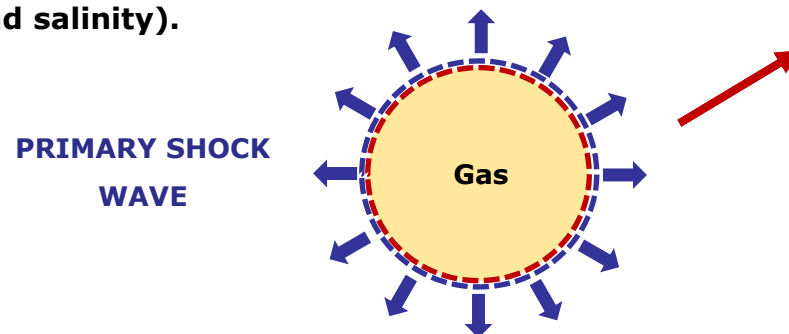


UNDERWATER EXPLOSION – PRIMARY SHOCK WAVE

- The explosion is activated by mean of a fuse (or detonator) giving the initial energy needed to ignite the detonation process.
- During the detonation process, a rapid transformation of the initial explosive reagent occurs into an expanding gas mass having high temperature and pressure (3000°C, 10³ MPa). The spherical front of chemical reaction represents the **DETONATION WAVE**, travelling at high speed (6000-9000 m/s) in the explosive mass domain. Detonation speed is **HIGHER** than the medium (i.e. explosive) sound speed.

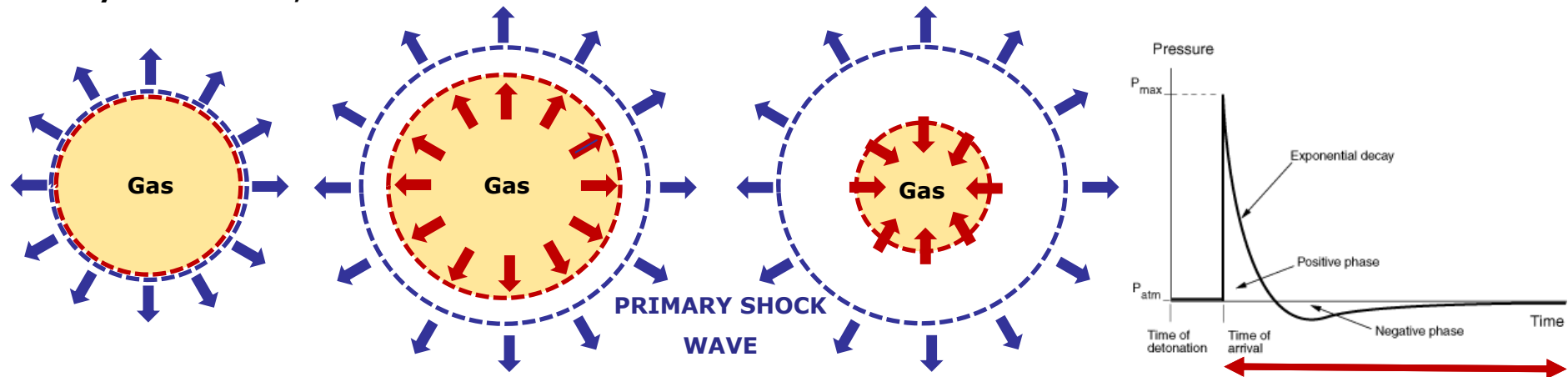


- Once the detonation wave reaches the limit of the explosive mass domain the explosion energy is transferred to the surrounding medium (seawater), giving rise to a **PRIMARY SHOCK WAVE** travelling in the water at the **SEAWATER SPEED OF SOUND** (about 1550 m/s in relation to water depth, temperature and salinity).

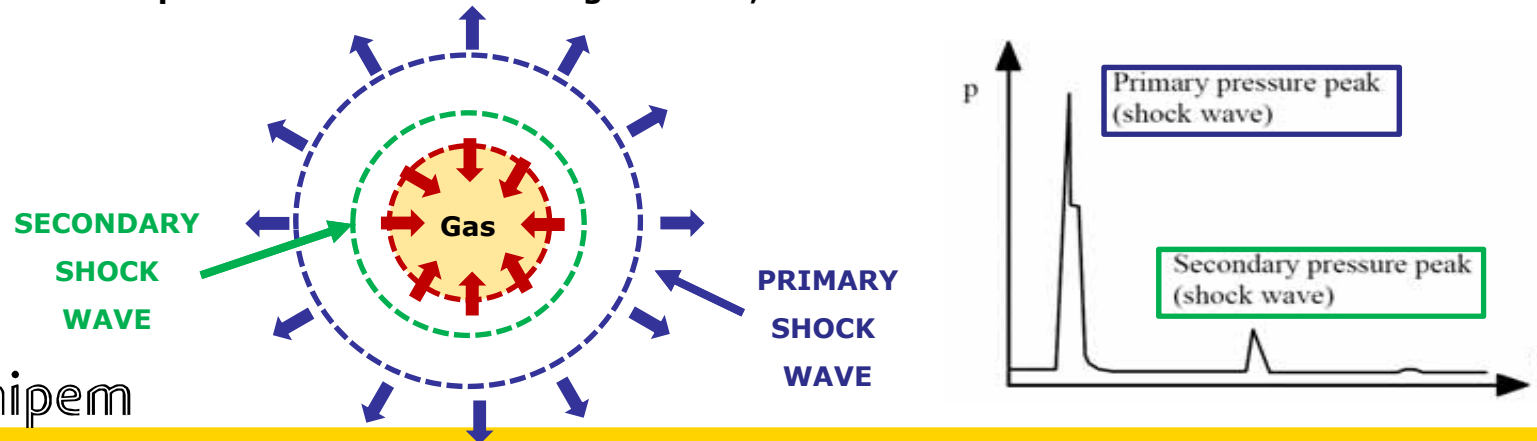


UNDERWATER EXPLOSION – SECONDARY SHOCK WAVE

- The initial pressure inside the gas sphere is much higher than the water hydrostatic pressure, causing the surrounding water to be subjected to a large outward acceleration due to the rapid **EXPANSION OF THE GAS BUBBLE** continuing also when the internal pressure is in equilibrium with the external hydrostatic one, due to the inertia of the accelerated water.



- When the outward movement of the gas bubble stops, the water viscoelasticity gives rise to an inward motion of the gas bubble spherical front, until the increasing pressure in the bubble reverses the motion. At this step a second shock wave is generated, so called **1st BUBBLE PULSE SHOCK WAVE**.



UNDERWATER EXPLOSION – SHOCK WAVE

- The viscoelasticity of the water and the behavior of the gas bubble give rise to a series of contraction and expansion cycles. At each cycle a pressure wave is released in the surrounding water. The entity of these waves is such negligible with respect to the INITIAL SHOCK WAVE and the 1st BUBBLE PULSE SHOCK WAVE.

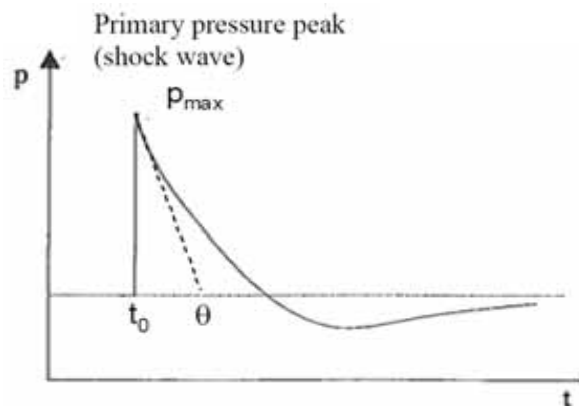


UNDERWATER EXPLOSION – EXPLOSION EFFECT

**CHARGE FAR FROM THE
STRUCTURE**

Distance $\gg R_{\text{bubble}}$

PRIMARY SHOCK WAVE

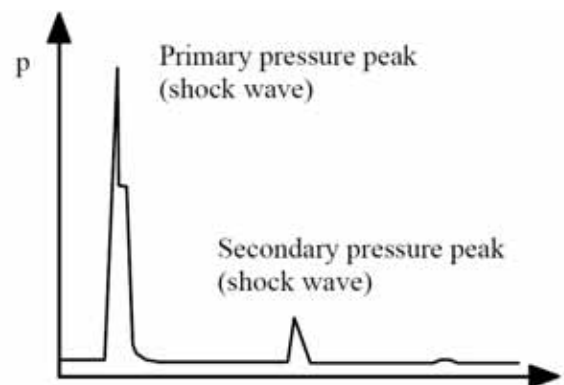


Target is far from the UXO, it interacts only with the first pressure wave. No secondary effects are experienced by the structure.

**CHARGE CLOSE TO THE
STRUCTURE**

Distance $> R_{\text{bubble}}$

**PRIMARY SHOCK WAVE +
SECONDARY SHOCK WAVES**

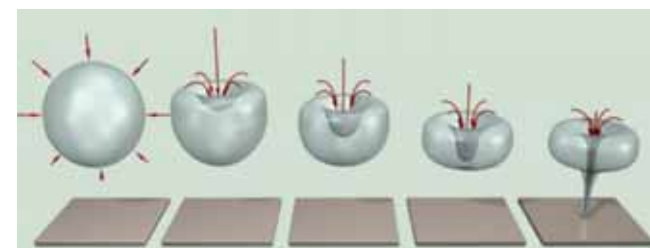


Target is close to the UXO and both primary and secondary pressure waves are experienced by the structure.

**CHARGE VERY CLOSE TO THE
STRUCTURE**

Distance $< R_{\text{bubble}}$

**PRIMARY SHOCK WAVE +
SECONDARY SHOCK WAVES +
WATER JETTING**



Target is very close to the UXO. It experiences all consequences of UNDEX and influences the bubble dynamics.

Water jetting phenomena are experienced.



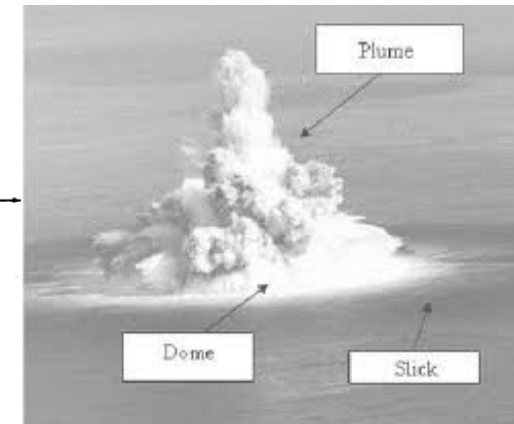
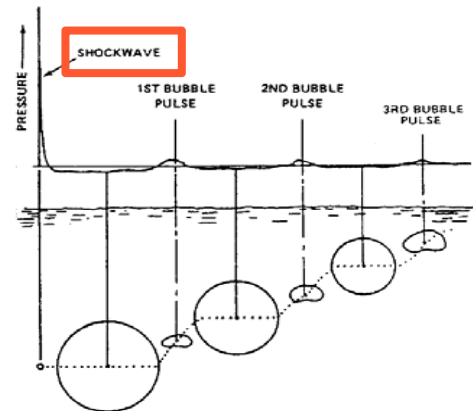
UNDERWATER EXPLOSION – SURFACES INTERACTION

FREE WATER EXPLOSION

FREE WATER UNDEX occurs when structure surfaces and other walls (seabed, sea surface, hulls, pipelines) are far from the explosive charge.

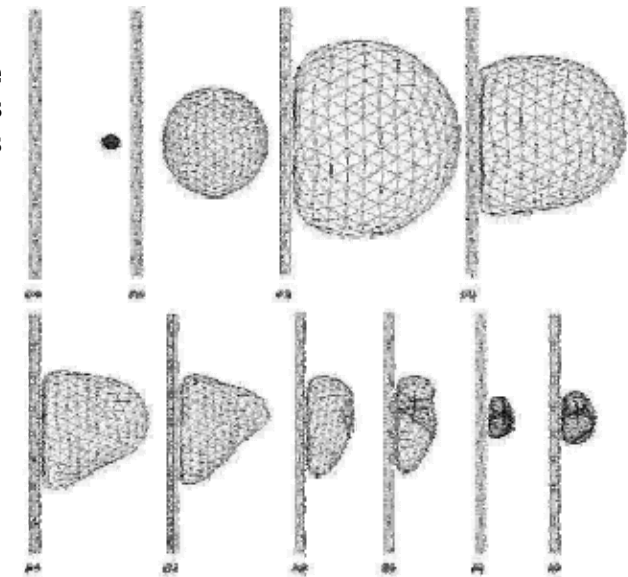
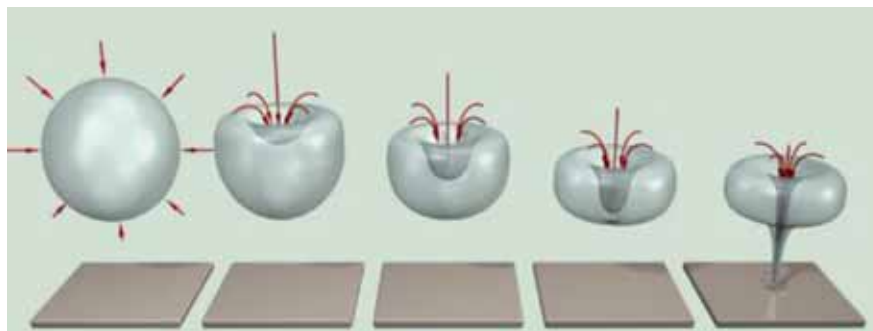
In this case no surface interactions arise, and the bubble evolves following the described process.

During the pulsation the bubble travels toward the sea surface. Surface **SPRAY DOME** can be observed in relation to the initial water depth of the charge.



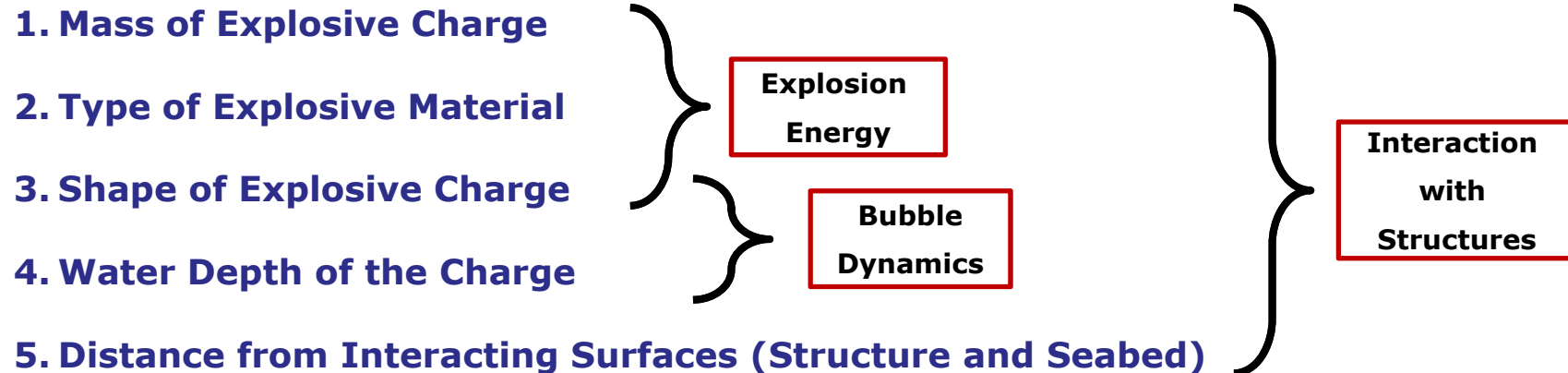
CLOSE SURFACES INTERACTION

The presence of a near wall deeply affects the bubble dynamics. The bubble is "attracted" by near surfaces. The pulsating bubble moves toward the surface and slams into it. A **HIGH SPEED WATER JETTING** hits the surface. This effect is also known as **BJERKNES FORCE**.



UNDERWATER EXPLOSION – PARAMETER EFFECT

UNDERwater EXplosion (UNDEX) is strictly affected by the following physical and geometrical parameters:



The previous parameters have influence on:

1. **Peak Pressure of Primary Shock Wave**
2. **Pressure of Secondary Shock Wave (1st Bubble Pulse)**
3. **Time History**
4. **Bubble Radius (R_{bubble})**
5. **Jetting Phenomena**



OBJECTIVE AND SCOPE OF WORK

The main objective of this study is to verify the structural integrity of a pipeline subject to the effects of the potential underwater explosion (UNDEX) of unexploded ordnances found in proximity of the pipeline.

The objective was achieved by using **FEM code ABAQUS**, and its specific capabilities/features for blasting and underwater explosion simulation.

The SoW includes:

- Pipeline **INTEGRITY CRITERIA** definition;
- Assessment of the **PROPAGATION IN WATER OF PRESSURE WAVES** induced by the underwater explosion of a spherical TNT charge, equivalent to the expected unexploded ordnance;
- Definition of **RELEVANT PIPELINE LOAD SCENARIOS** induced by the interaction between the pressure wave and the pipeline shell;
- Characterisation of the **PIPELINE DYNAMIC RESPONSE**, in terms of activated local and global deformation modes;
- Pipeline response analysis and integrity assessment: definition of a relationship between the weight of the spherical TNT charge and the **MINIMUM DISTANCE** from the pipeline.



PIPELINE INTEGRITY CRITERIA

STRESS BASED CRITERION

No damage experienced by pipeline wall due to the underwater explosion. The **MAXIMUM VON MISES STRESS** shall be less than **96% SMYS** (namely 432MPa).

SERVICEABILITY LIMIT STATES (SLS)

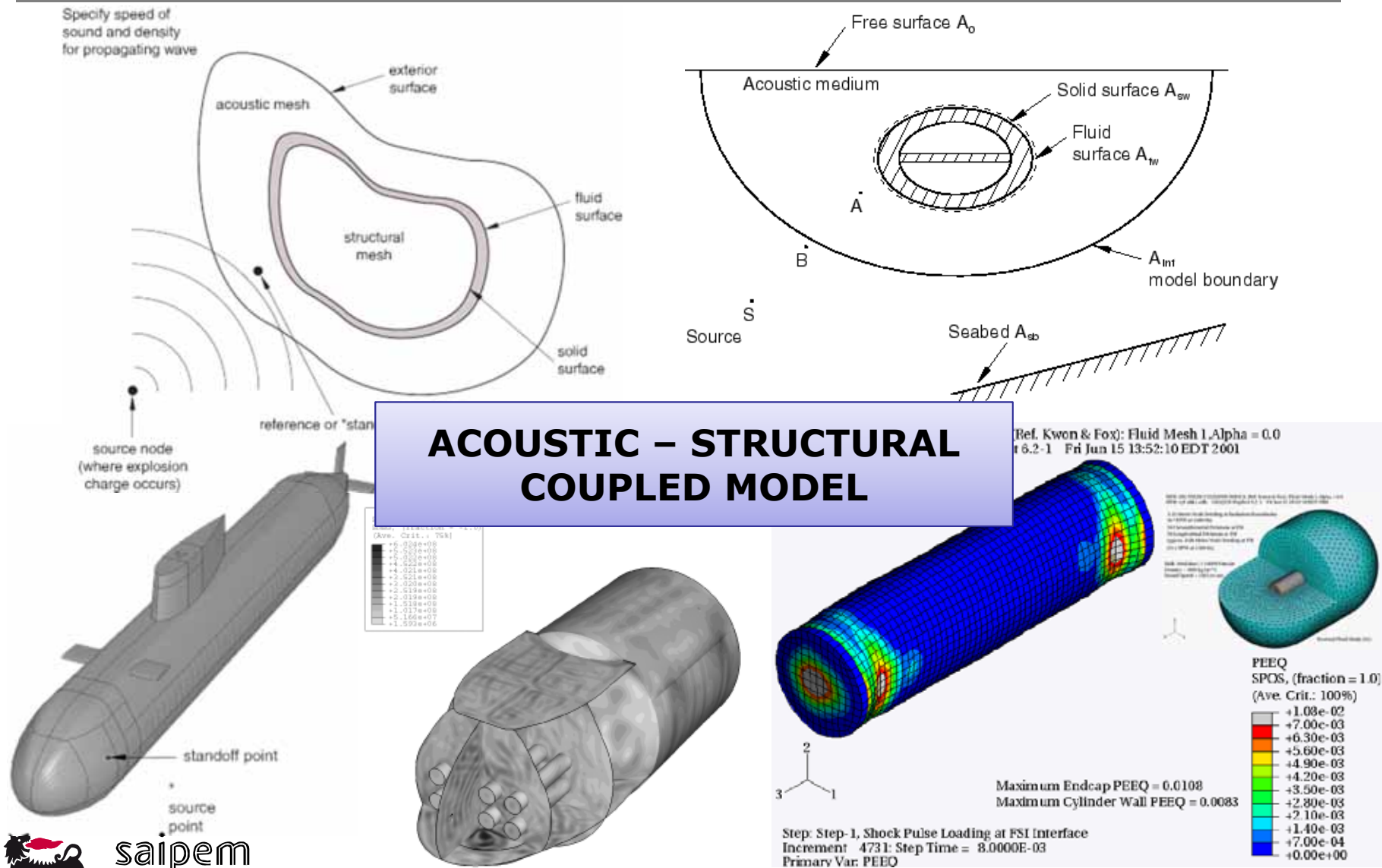
- **OVALIZATION BASED CRITERION:** in accordance to DNV OS-F101, the pipeline shall not be subject to excessive ovalization. The residual **FLATTENING** is not to exceed **3.0%**.
- **DENT BASED CRITERION:** in accordance to DNV-RP-F107 **DENT** to diameter ratio shall be limited to **5.0%**.

ULTIMATE LIMIT STATE (ULS)

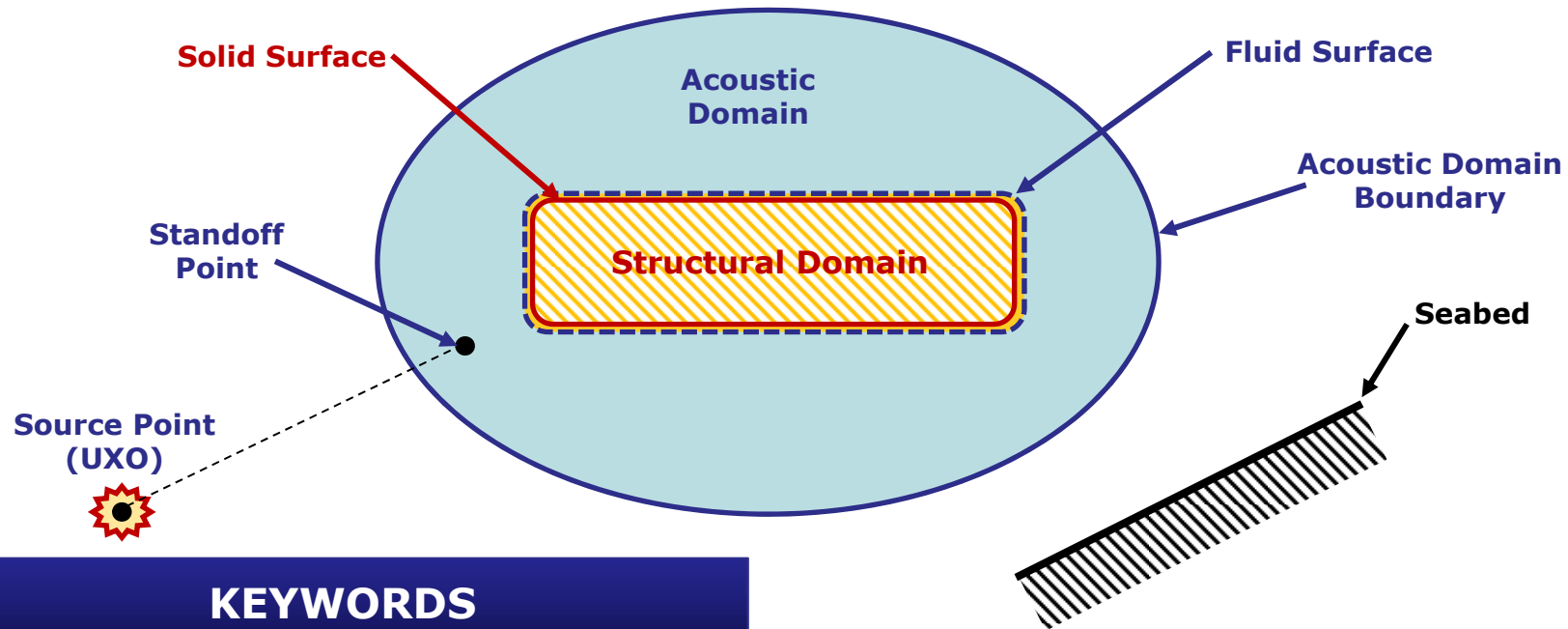
The pipe wall may experience **SIGNIFICANT PLASTIC STRAINS**, but the pipe wall tearing or a gas leakage shall not appear (corresponding to a **MAXIMUM EQUIVALENT PLASTIC STRAIN** equal to the uniform elongation limit = **10%**).



UNDEX MODELING IN ABAQUS



ABAQUS UNDEX MODELING – ACOUSTIC-STRUCTURAL COUPLING



KEYWORDS

***INCIDENT WAVE INTERACTION PROPERTY**

Definition of fluid properties for **ACOUSTIC DECAY** calculation of incident wave.

***INCIDENT WAVE INTERACTION**

Definition of incident wave loading, hit surface and **SOURCE** and **STANDOFF** points.

***UNDEX CHARGE PROPERTY**

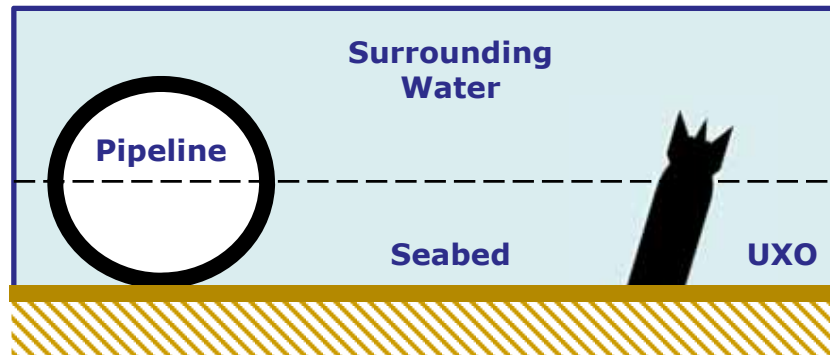
Definition of **UXO** charge **PROPERTIES**.

***SIMPEDANCE**

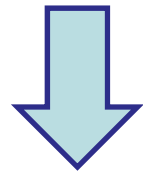
Definition of surface reflection properties.



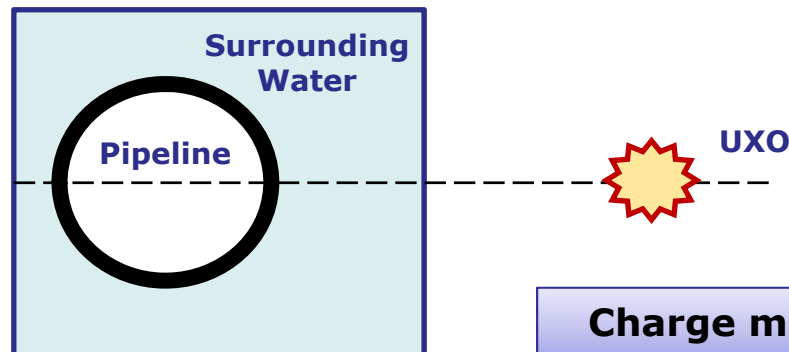
PIPELINE-UXO INTERACTION – MODEL ASSUMPTIONS



Charge mass = M



- Seabed surface has been considered **PERFECTLY REFLECTIVE**. This is a **CONSERVATIVE** assumption since no explosion energy amount is absorbed by soil (e.g. for a crater formation). All explosive power diffuses through the acoustic medium and hits the pipeline.
- For this reason in the FEM model the assumed mass charge has been **DOUBLED** with respect to the real mass of explosive charge.
- The explosive mass has been assumed as a **POINT SOURCE**, and the wave propagation has been modeled as **SPHERICAL**.
- The TNT charge has been modelled considering the **GEERS-HUNTER** model.



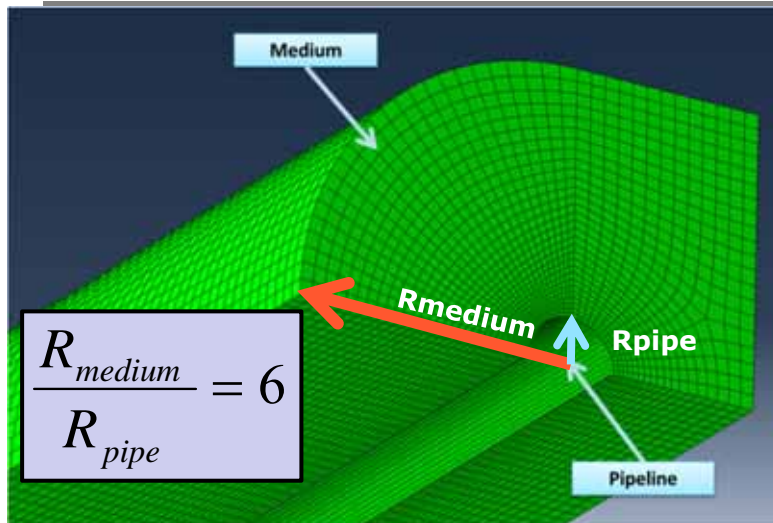
Charge mass = $2M$

Geers-Hunter model

Charge constant K	5.97e+07
Charge constant k	8.83e-05
Similitude spatial exponent A	0.13
Similitude temporal exponent B	0.18
Charge constant K_c	1.05e+09
Ratio of specific heats for explosion gas	1.27
Charge material density	1654

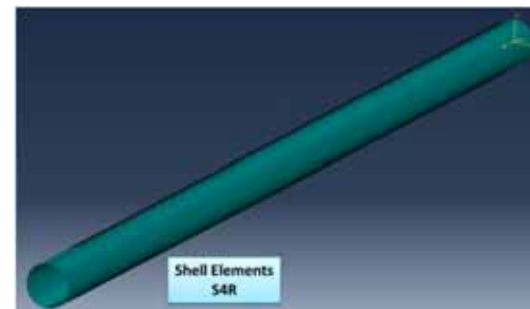
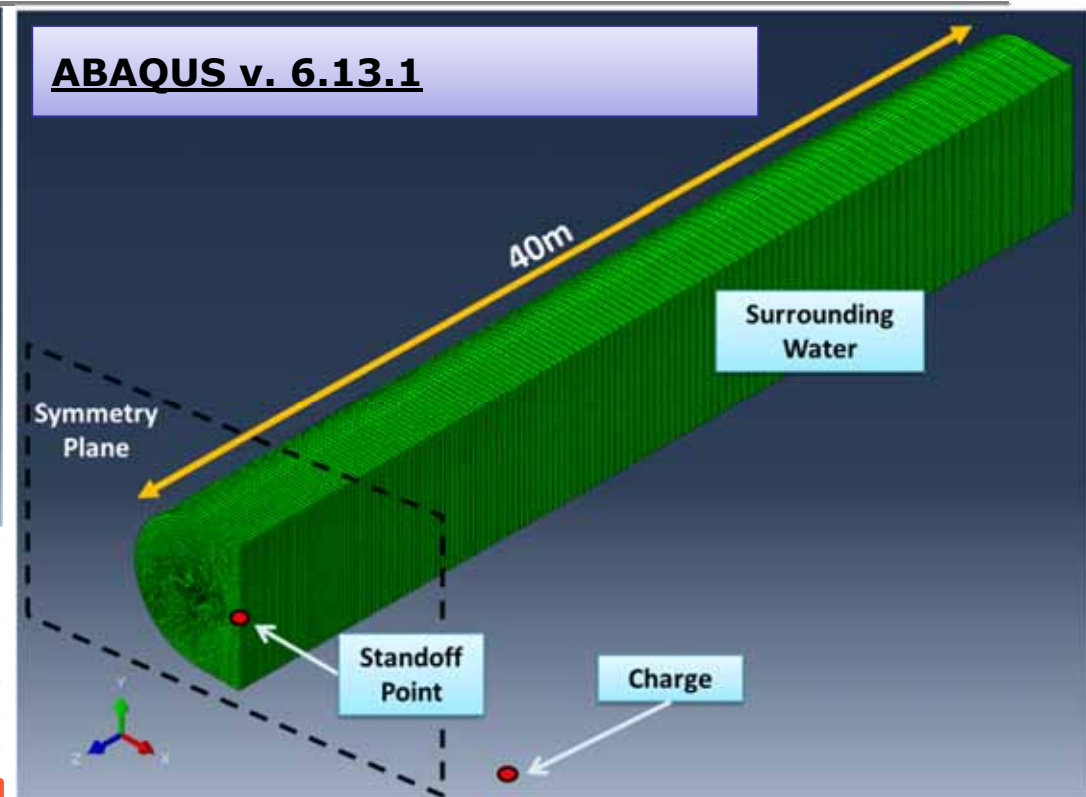


PIPELINE-UXO INTERACTION – FE MODEL



Added Mass for N=1 Translation Mode of an Infinite Cylinder (Fluid Between Concentric Cylinders – Blevins ¹)	
Cylinder Radius Ratio (R_o/R_i)	Added Mass Ratio (External Boundary/Infinite Domain)
1.5	2.600
2.0	1.667
4.0	1.133
6.0	1.057
8.0	1.032
16.0	1.008
24.0	1.004

- PIPELINE – **S4R** SHELL ELEMENTS
- WATER – **AC3D8R** ACOUSTIC BRICK ELEMENTS



MODEL VALIDATION – ANALYTICAL APPROACH

- INITIAL SHOCK WAVE (due to Detonation wave)**

Cole, R.H.: "Underwater Explosions",
Dover Publications Inc., N.Y. , 1965

$$P = P_{\max}(D, W) \exp \left[- \frac{t - t_0}{\mathcal{G}(D, W)} \right]$$

D = Charge Distance
W = Charge Weight

- BUBBLE MOTION EQUATION (Rayleigh-Plesset)**

$$\rho \left[R \ddot{R} + \frac{3}{2} \dot{R}^2 \right] = P_B(t) - P_{\infty}$$

$$R_{\min} = 8.24 \frac{W^{5/9}}{(wd + 10.3)^{11/9}} + 0.007 \cdot W^{5/16} \quad R_{\max} = 3.36 \sqrt[3]{\frac{W}{wd + 10}}$$

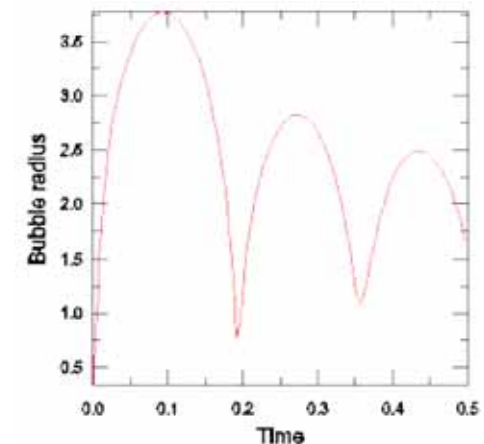
$$T_{\text{bubble}} = 2.11 \frac{W^{1/3}}{(wd + 10)^{5/6}}$$

$$P_{\text{bubble}, \max} = 1.9 \frac{3}{4\pi} \frac{1}{D} \frac{W}{R_{\max}^2} \left(1 - \frac{0.1581 \cdot W^{0.25}}{R_{\max}^{0.75}} \right)$$

D = Charge Distance
W = Charge Weight
wd = water depth

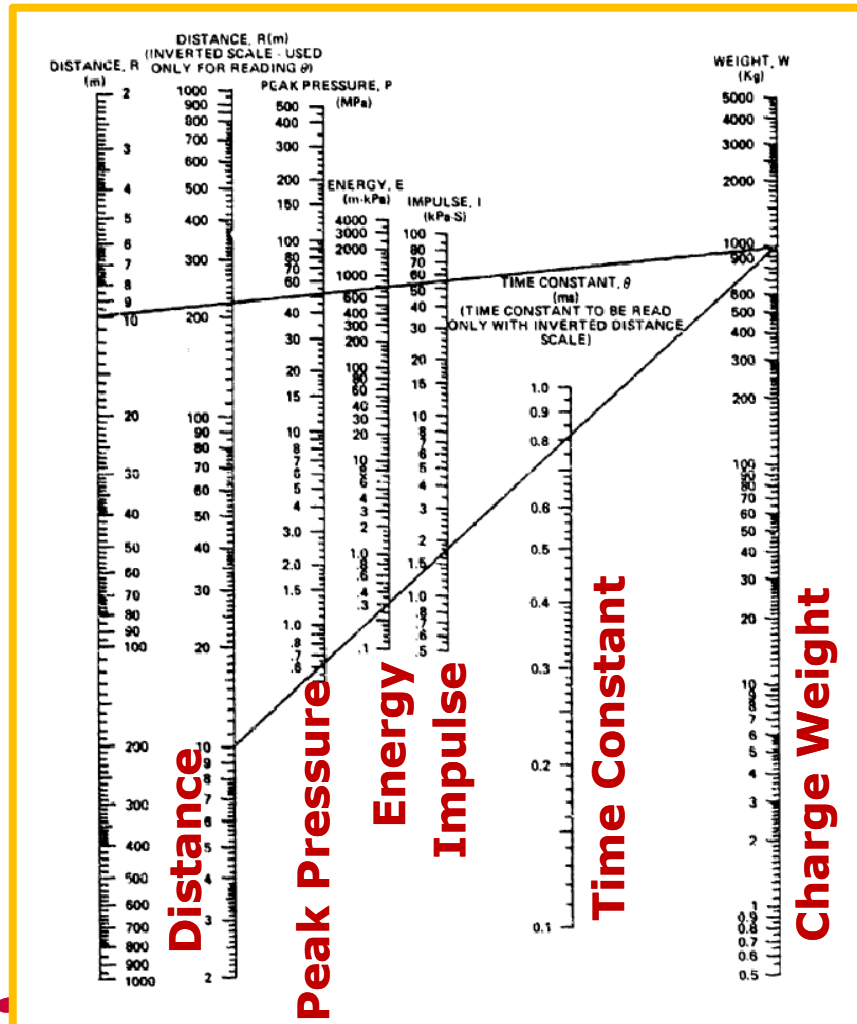
M.M. Swisdak Jr, "Explosion effects and Properties: Part II – Explosion Effects in Water", Naval Surface Weapon Center, 1978.

Petralia S., 2000, Explosivistic Compendium, Mariperman La Spezia.

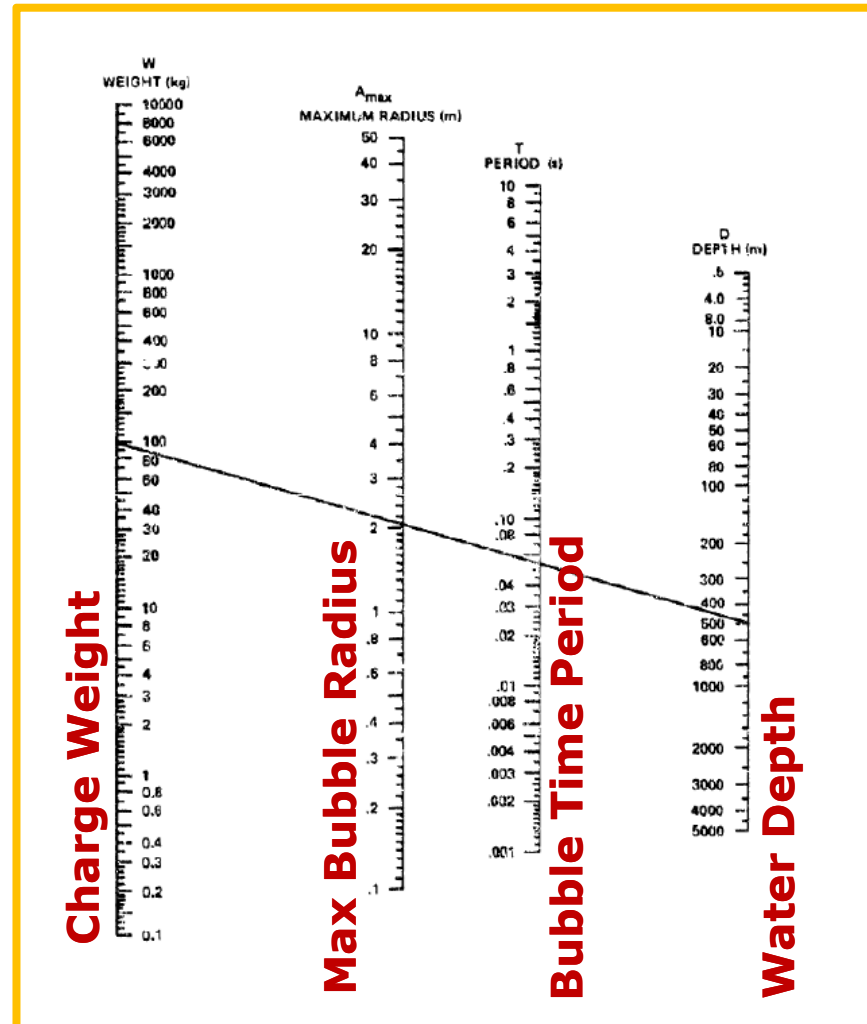


MODEL VALIDATION – NOMOGRAPH

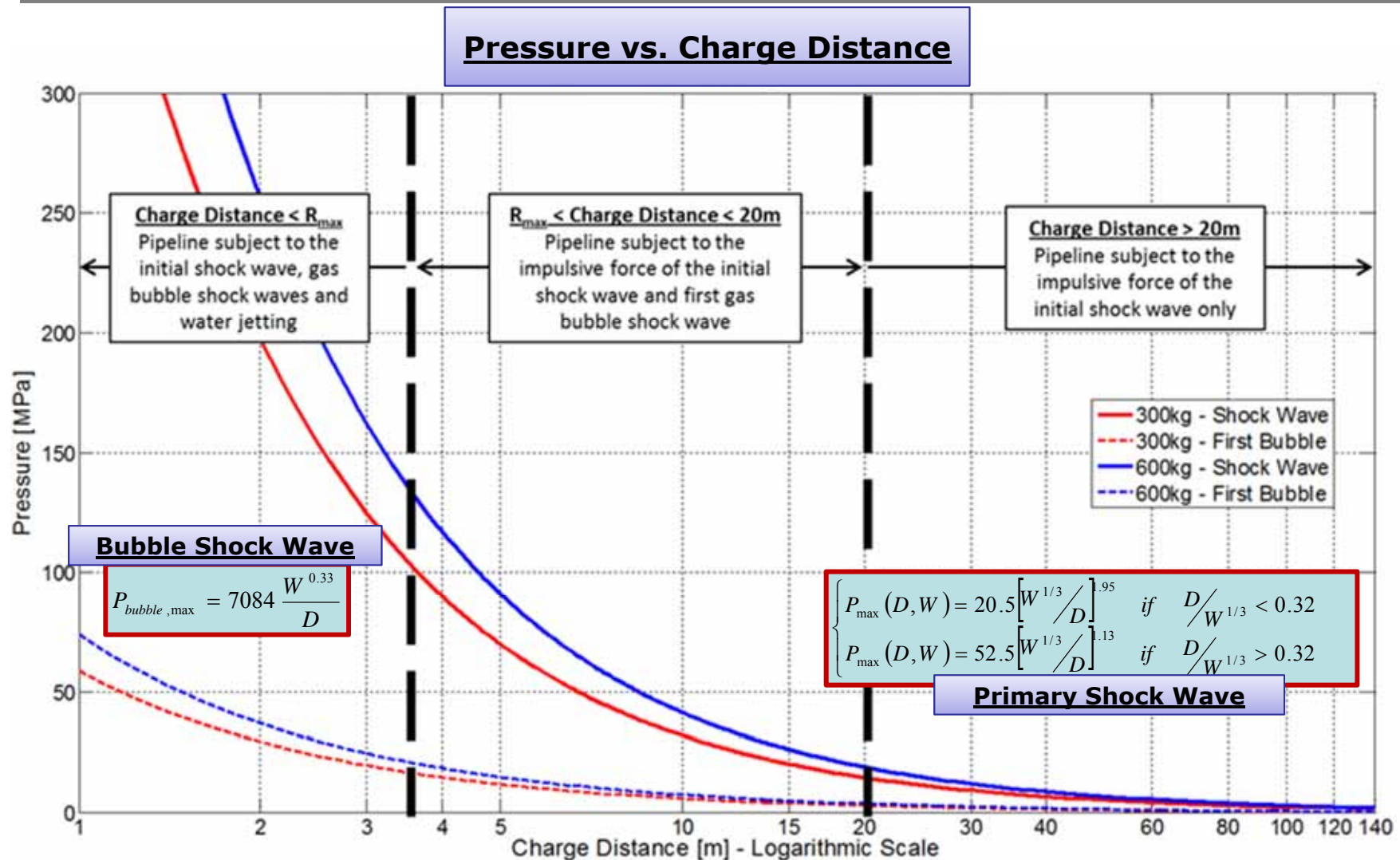
NOMOGRAPH FOR INITIAL SHOCK WAVE EVALUATION



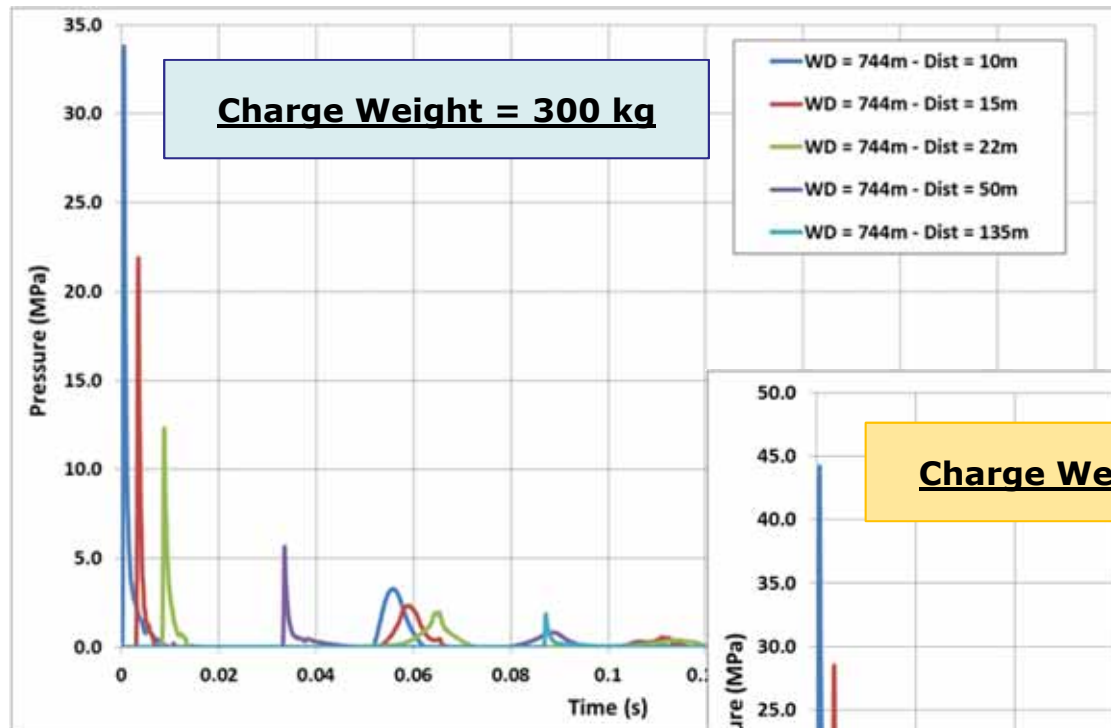
NOMOGRAPH FOR BUBBLE DYNAMICS CHARACTERIZATION



UNDEX CHARACTERIZATION AND FE MODEL VALIDATION



UNDEX CHARACTERIZATION AND FE MODEL VALIDATION



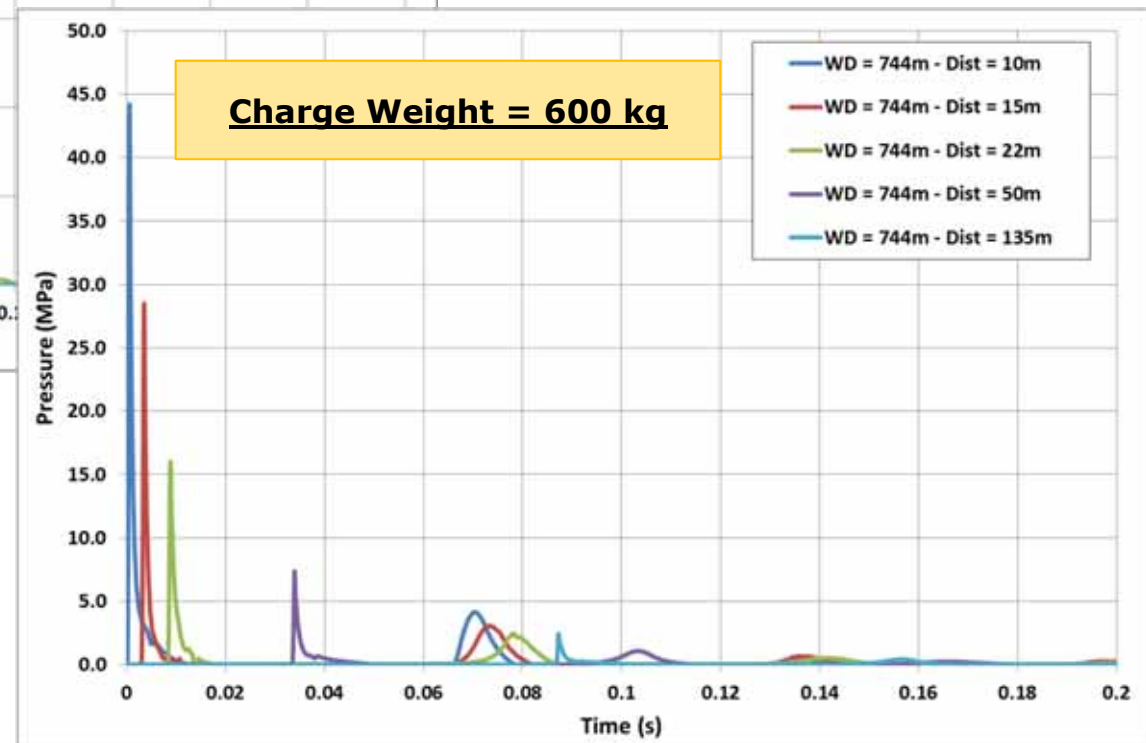
Shock Pressure Time History

$$P = P_{\max}(D, W) \exp \left[-\frac{t - t_0}{g(D, W)} \right]$$

Cole

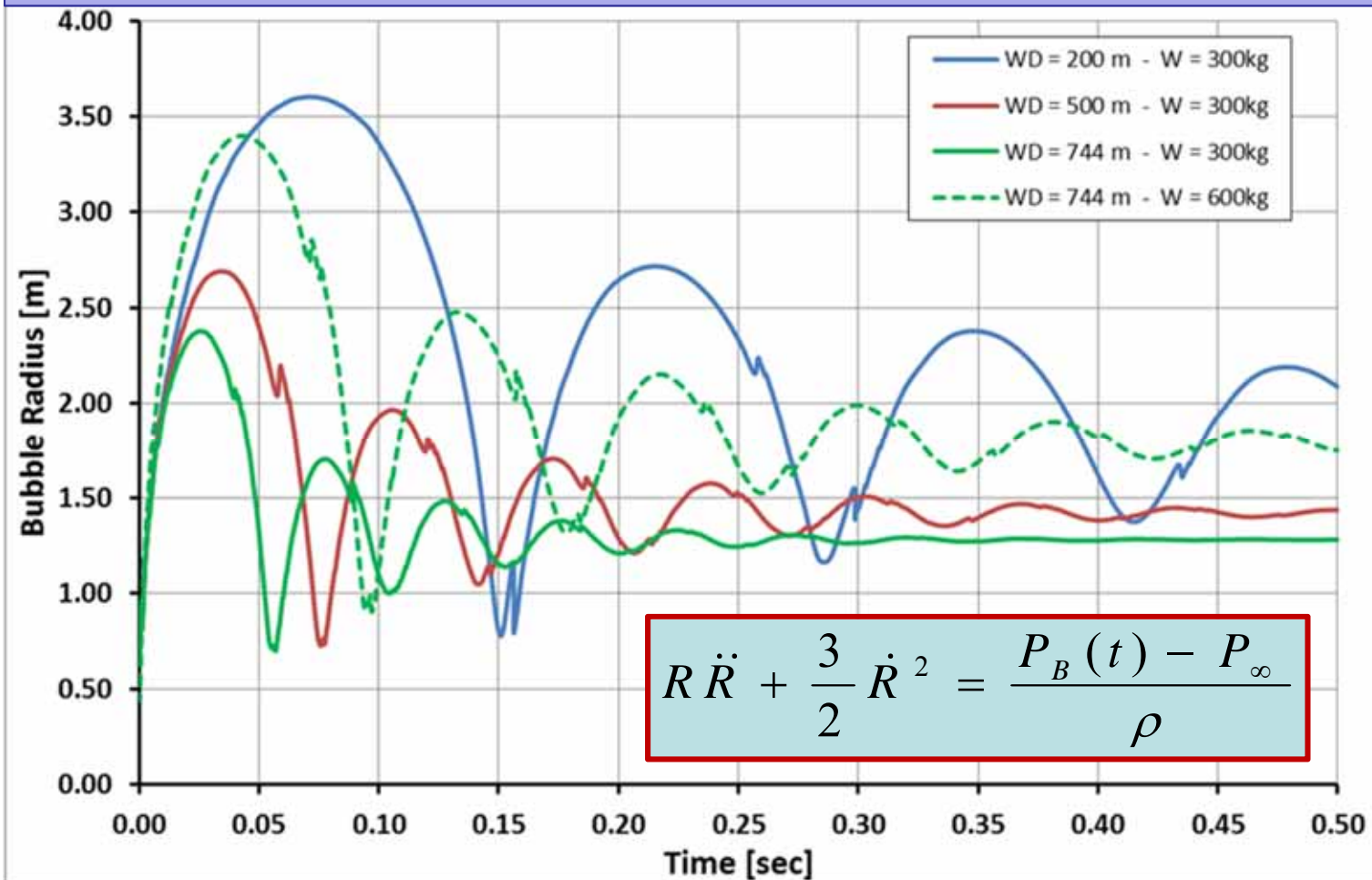
$$\begin{cases} P_{\max}(D, W) = 20.5 \left[\frac{W^{1/3}}{D} \right]^{.95} & \text{if } \frac{D}{W^{1/3}} < 0.32 \\ P_{\max}(D, W) = 52.5 \left[\frac{W^{1/3}}{D} \right]^{.13} & \text{if } \frac{D}{W^{1/3}} > 0.32 \end{cases}$$

$$\begin{cases} g(D, W) = 0.0001048 \cdot W^{1/3} \left[\frac{D}{W^{1/3}} \right]^{0.35} & \text{if } \frac{D}{W^{1/3}} < 2.70 \\ g(D, W) = 0.0001163 \cdot W^{1/3} \left[\frac{D}{W^{1/3}} \right]^{0.25} & \text{if } \frac{D}{W^{1/3}} > 2.70 \end{cases}$$

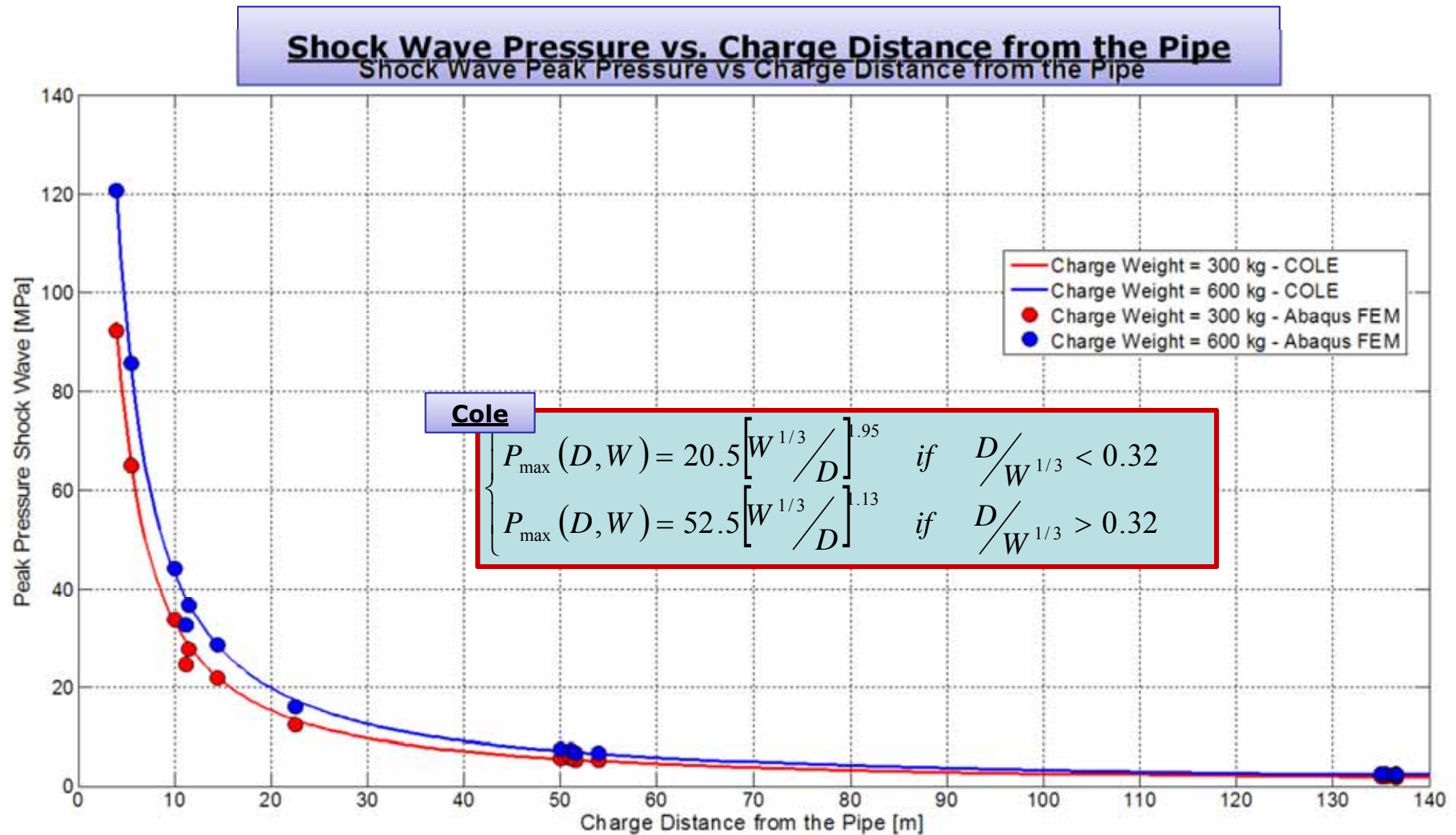


UNDEX CHARACTERIZATION AND FE MODEL VALIDATION

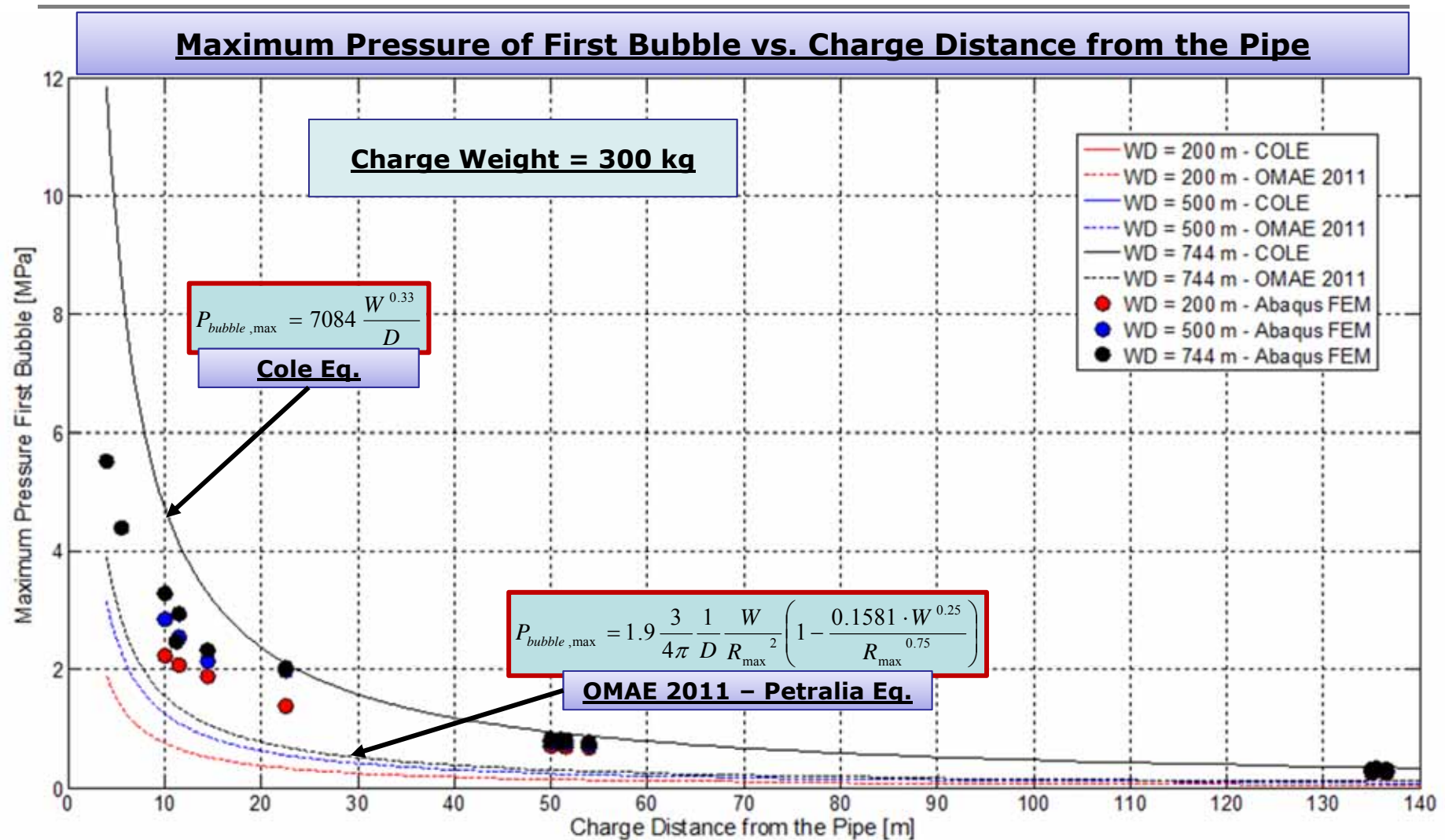
Time History of the Bubble Radius as a Function of the Water Depth and Charge Weight



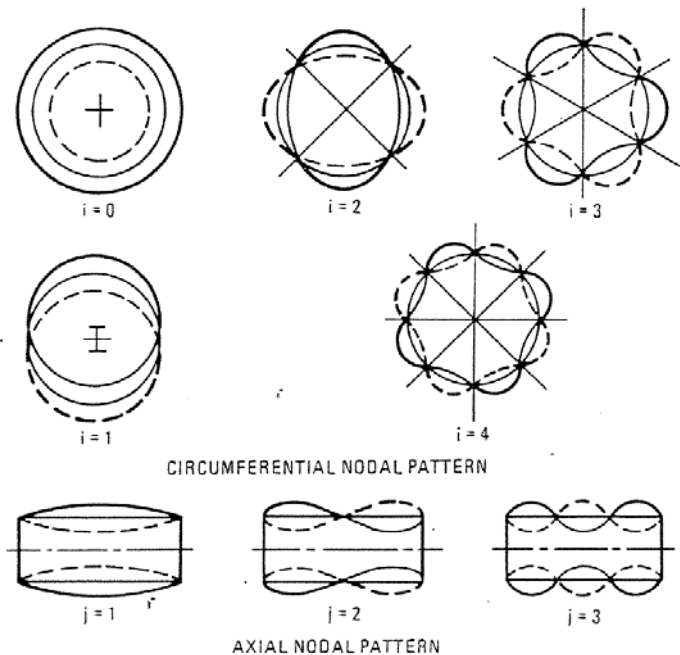
INDEX CHARACTERIZATION AND FE MODEL VALIDATION



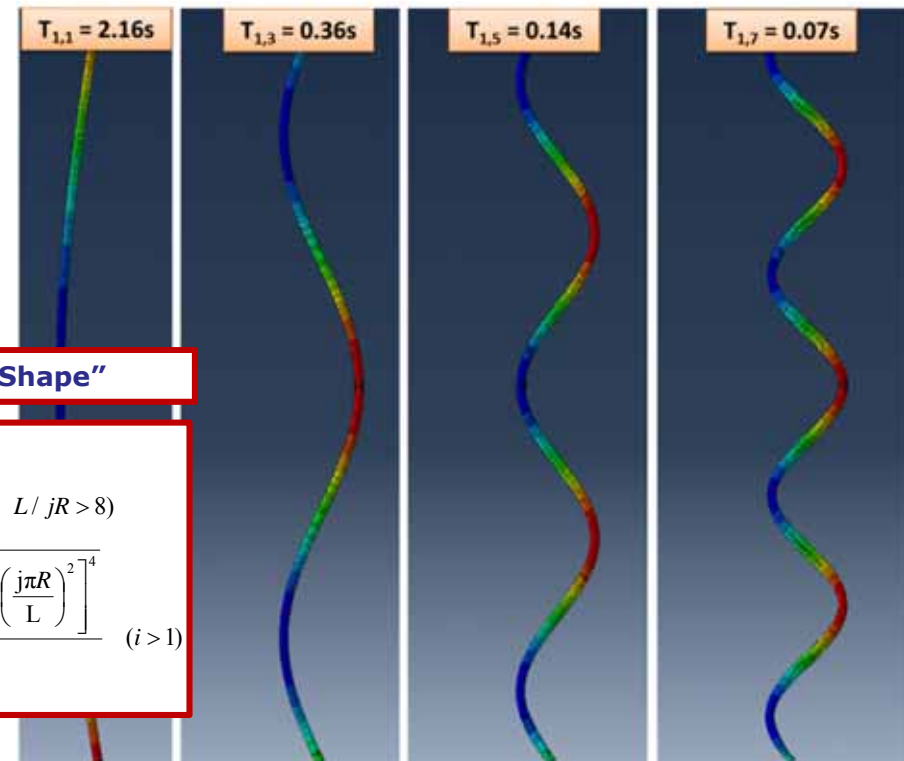
INDEX CHARACTERIZATION AND FE MODEL VALIDATION



PIPELINE DYNAMIC RESPONSE – FEM VS. ANALYTICAL RESULTS



T_{ij} [s]	1,1	1,3	1,5	1,7
FEM	2.16	0.36	0.14	0.07
Analytic	2.39	0.26	0.10	0.05



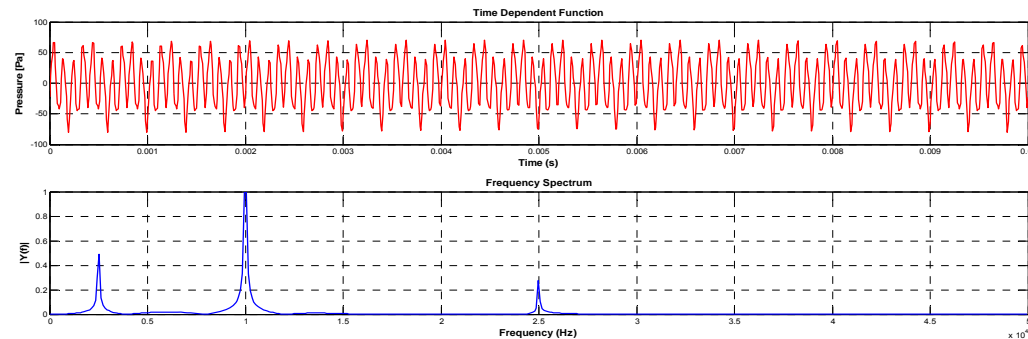
R.D. Blevins: "Formulas for Natural Frequency and Mode Shape"

$$\begin{cases} u_{i,j}(x, \theta) = A \cdot \cos(i \cdot \theta) \cdot \cos\left(\frac{j\pi R}{L}\right) \\ v_{i,j}(x, \theta) = B \cdot \sin(i \cdot \theta) \cdot \sin\left(\frac{j\pi R}{L}\right) \\ w_{i,j}(x, \theta) = C \cdot \cos(i \cdot \theta) \cdot \sin\left(\frac{j\pi R}{L}\right) \end{cases}, (i \geq 1)$$

$$\begin{aligned} \lambda_{0,j} &= 1 \quad (i=0 \text{ and } L/jR > 8) \\ \lambda_{1,j} &= \frac{j^2 \pi^2 (1-\nu^2)^{1/2}}{\sqrt{2}} \frac{R^2}{L^2} \quad (i=1 \text{ and } L/jR > 8) \\ \lambda_{1,j} &= \frac{\sqrt{(1-\nu^2) \left(\left(\frac{j\pi R}{L} \right)^4 + \left(\frac{thk^2}{12R^2} \right) \right) i^2 + \left(\frac{j\pi R}{L} \right)^2}}{i^2 + \left(\frac{j\pi R}{L} \right)^2} \quad (i > 1) \end{aligned}$$

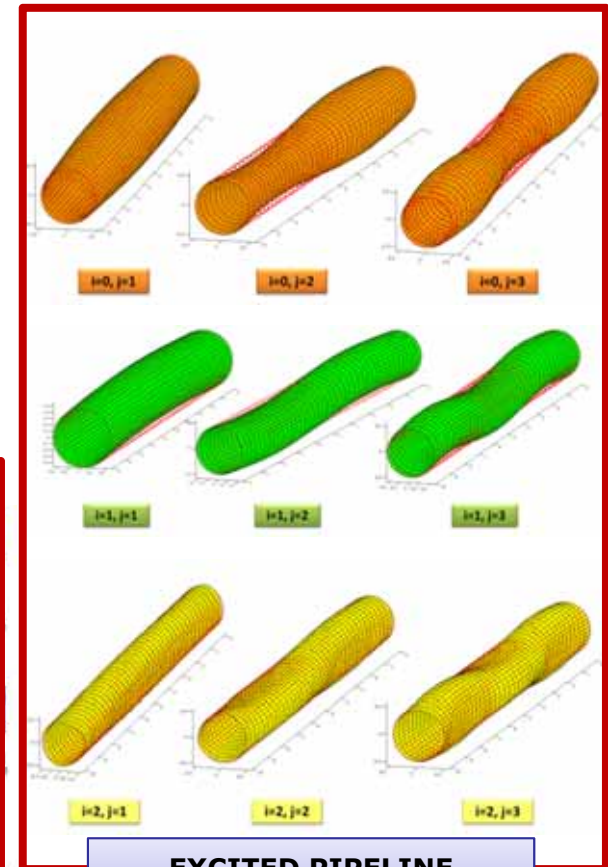
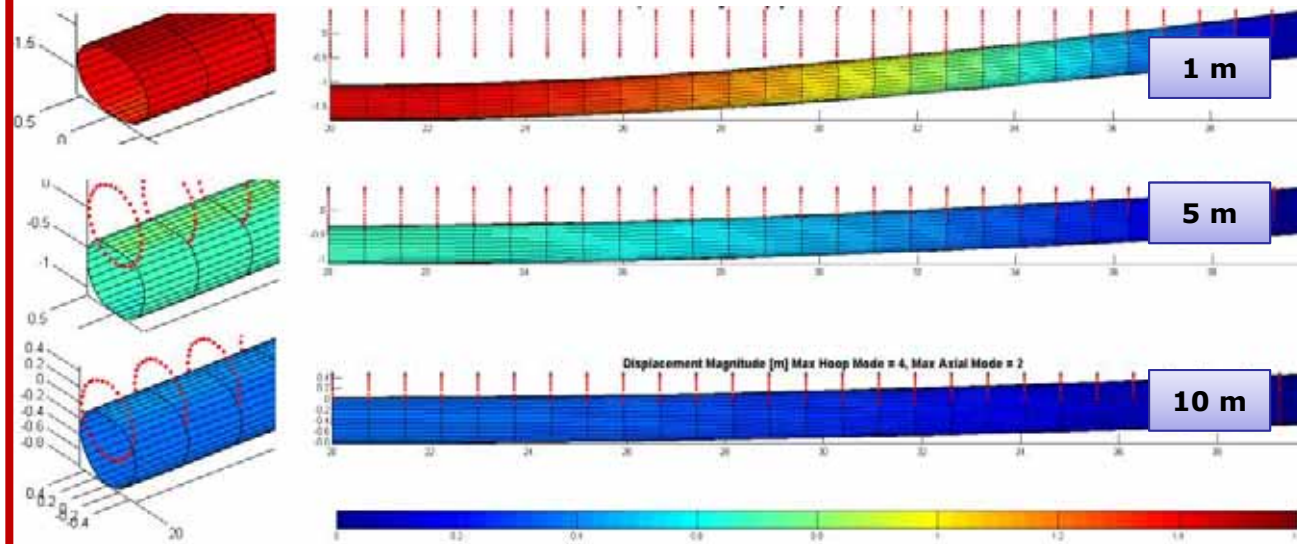


PIPELINE DYNAMIC RESPONSE – EXCITED NATURAL MODES



FREQUENCY SPECTRUM OF SHOCK WAVE (FFT)

PIPELINE RESPONSE



EXCITED PIPELINE NATURAL MODES

APPLICATION – PIPELINE BASIC DATA

Property	Units	Offshore 36" Pipe
Internal Diameter (Constant)	mm	871.0
Steel Wall Thickness	mm	34.0
Internal Coating	μm	60 to 110
Corrosion Allowance	mm	0.0
Manufacturing Method	-	UOE
Welding process	-	SAW
Fabrication Thickness Tolerance (body)	mm	+1.0 -1.0
Out of Roundness (body)	%/mm	1.0/10

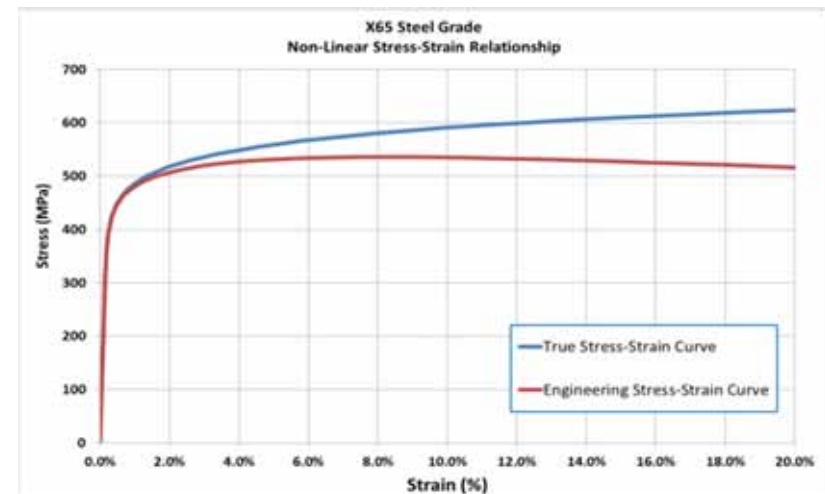
Property	Units	Offshore 36" Pipe
Material Grade	-	L450
Specified Minimum Yield Stress at 20°C	MPa	450
Specified Minimum Tensile Stress at 20°C	MPa	535
Density	kg/m^3	7850
Coefficient of linear thermal expansion	$^{\circ}\text{C}^{-1}$	1.16×10^{-5}
Young's Modulus	MPa	207×10^3
Poisson's Ratio	-	0.3

Pipeline Water Depth

- **WD = 744 m**

Two scenarios have been analysed:

- **$P_{int} = 0$**
- **$P_{int} = P_{des} = 145 \text{ barg}$**



APPLICATION – ORDNANCE BASIC DATA

MASS OF EXPLOSIVE CHARGE

Warhead Mass (by survey) = 300 kg

Warhead Mass (Safety Factor 2) = 600 kg

WARHEAD EXPLOSIVE TYPE

Trinitrotoluene (TNT)

WARHEAD SHAPE AND DIMENSIONS

Torpedo – Spherical Warhead

Assumed as Point Source in FE Model



Geers-Hunter model

Charge constant K	5.97e+07
Charge constant k	8.83e-05
Similitude spatial exponent A	0.13
Similitude temporal exponent B	0.18
Charge constant K_c	1.05e+09
Ratio of specific heats for explosion gas	1.27
Charge material density	1654



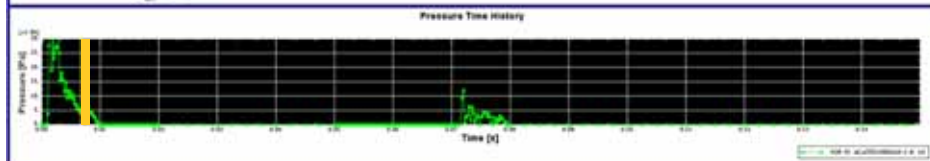
FEM STUDY RESULTS – UNDERWATER EXPLOSION

Charge Weight = 600 kg
Water Depth = 744 m
Charge Distance = 4 m

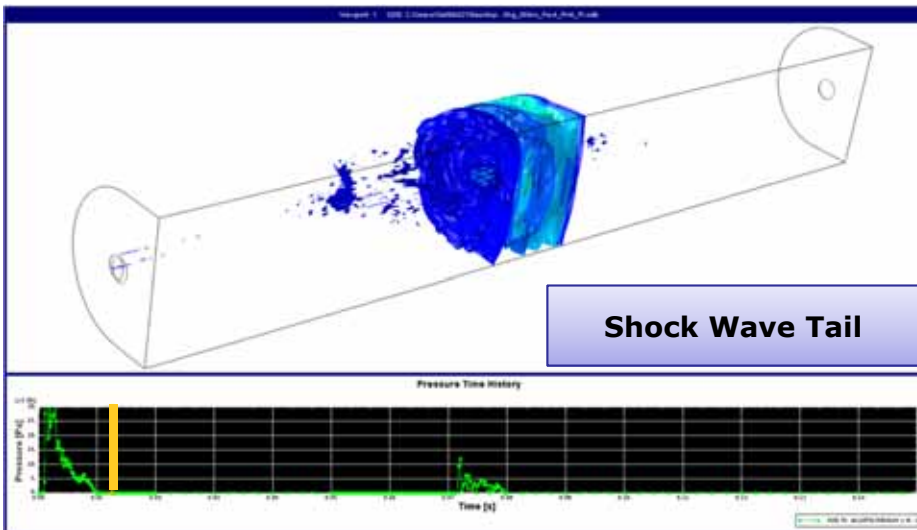
Primary Shock Wave



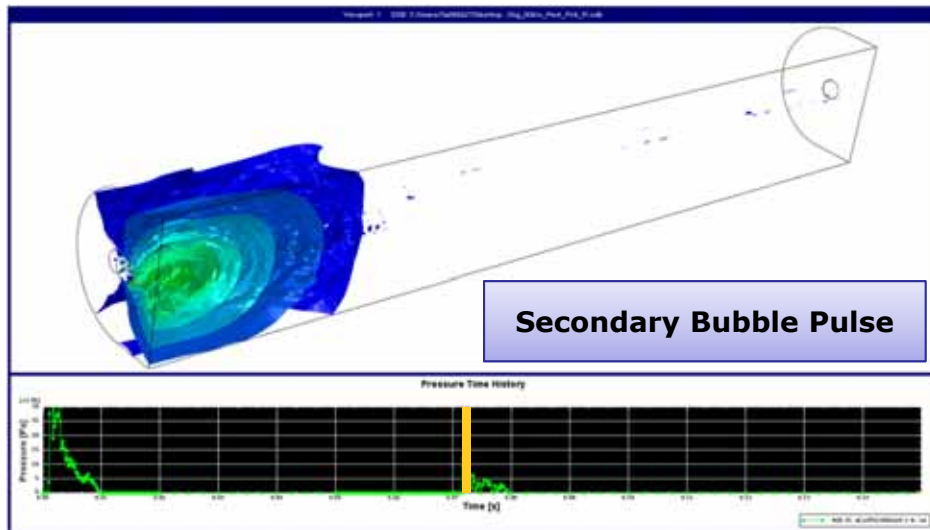
Wave Propagation



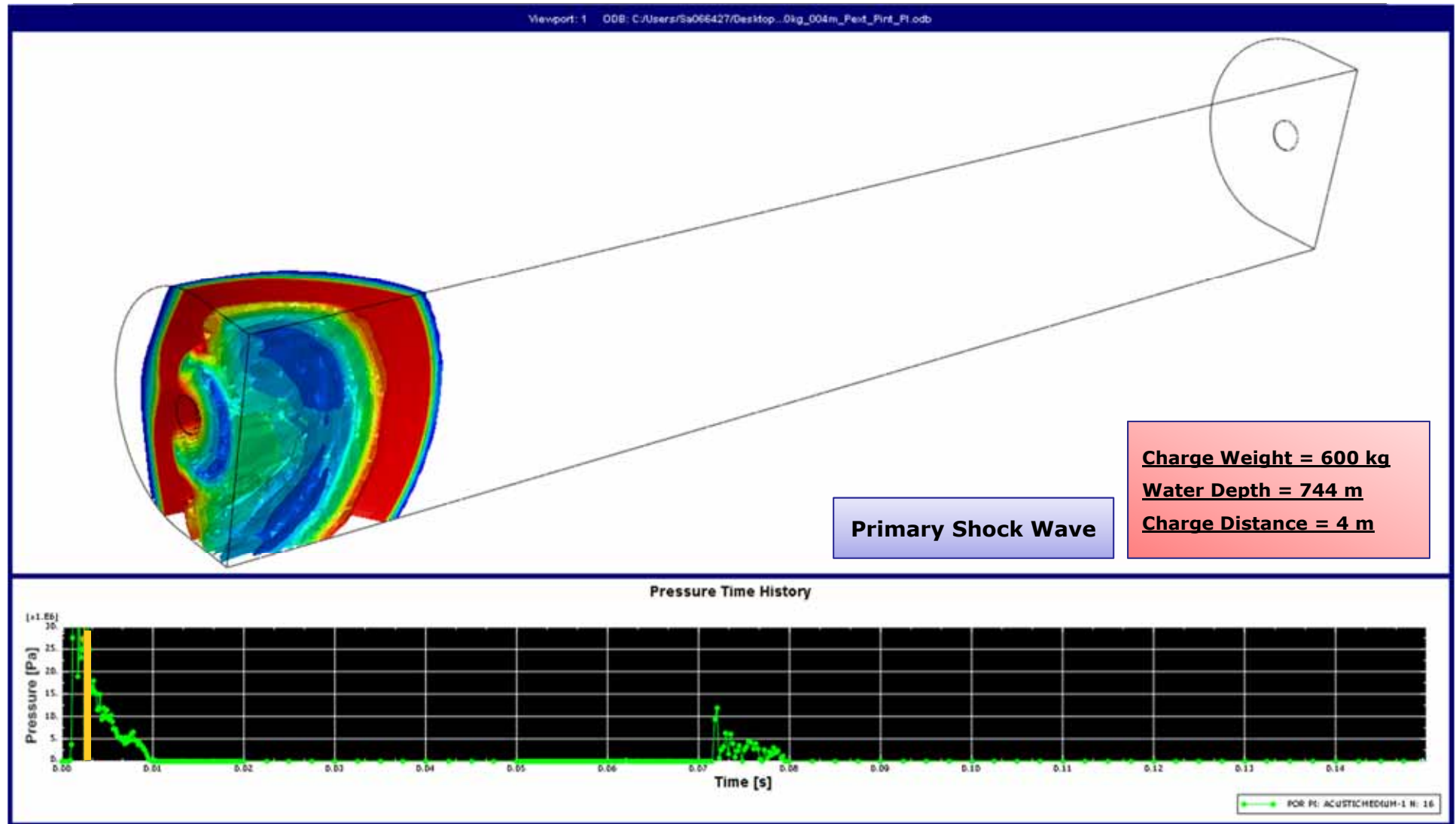
Shock Wave Tail



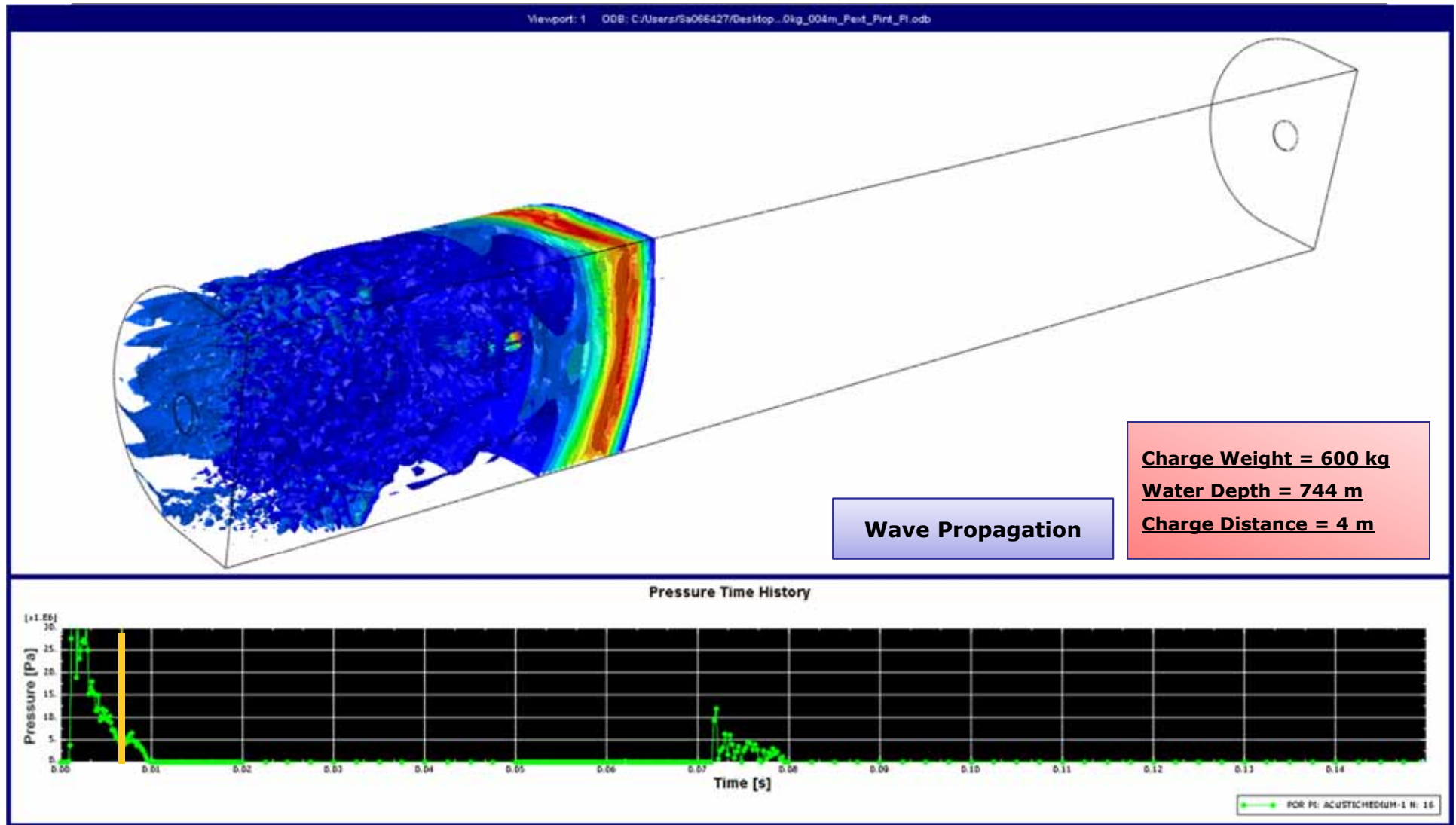
Secondary Bubble Pulse



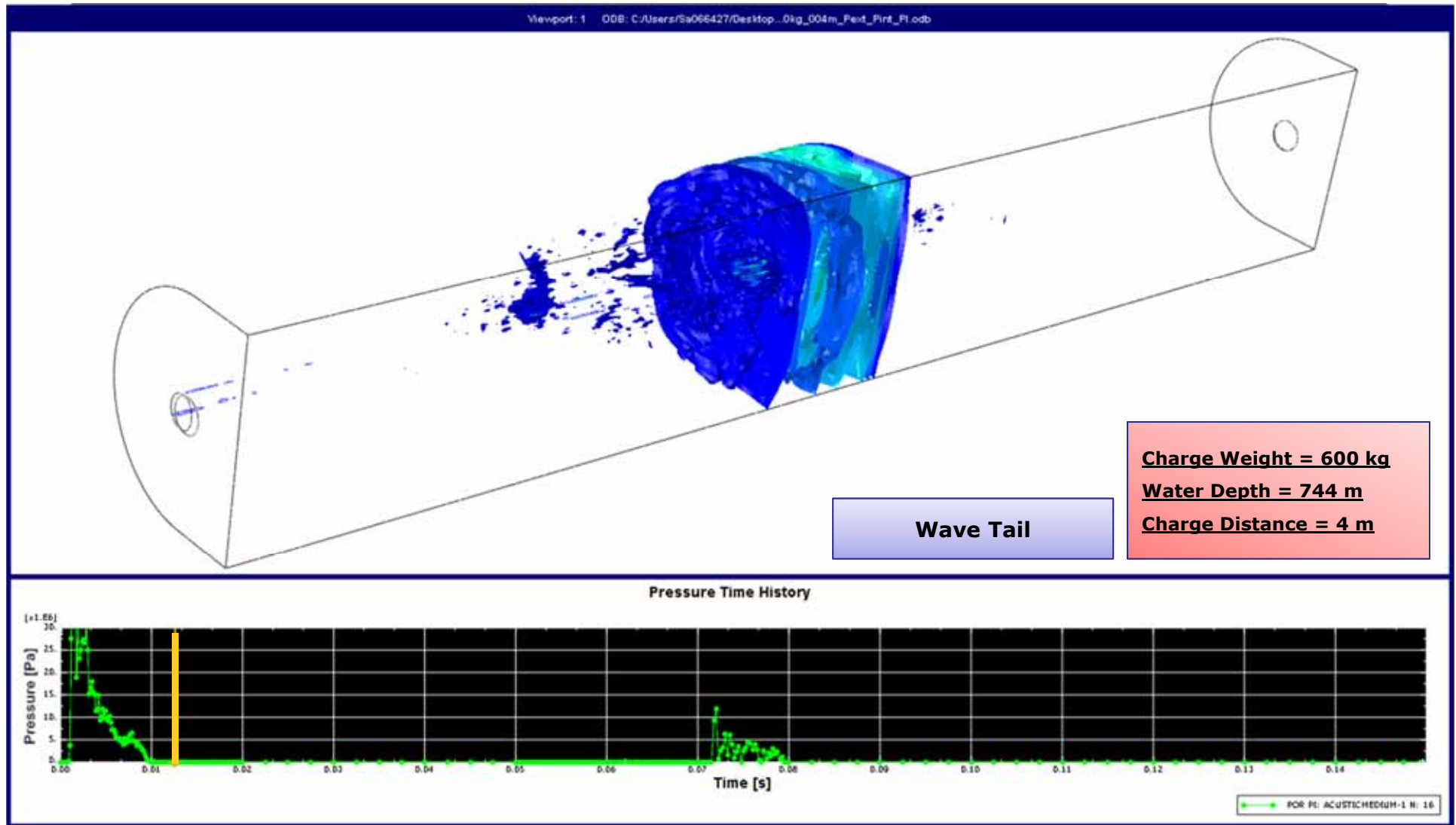
FEM STUDY RESULTS – UNDERWATER EXPLOSION



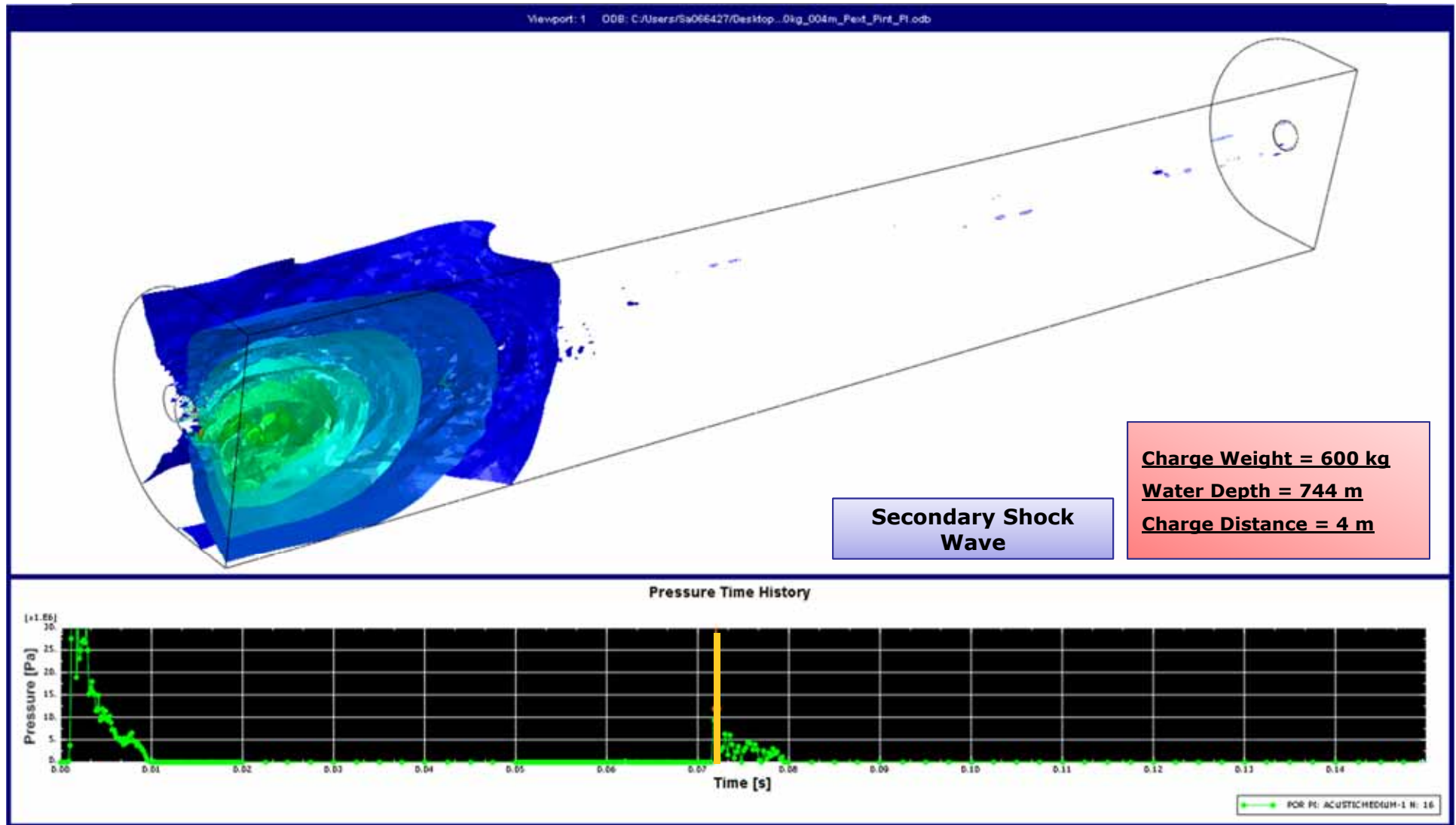
FEM STUDY RESULTS – UNDERWATER EXPLOSION



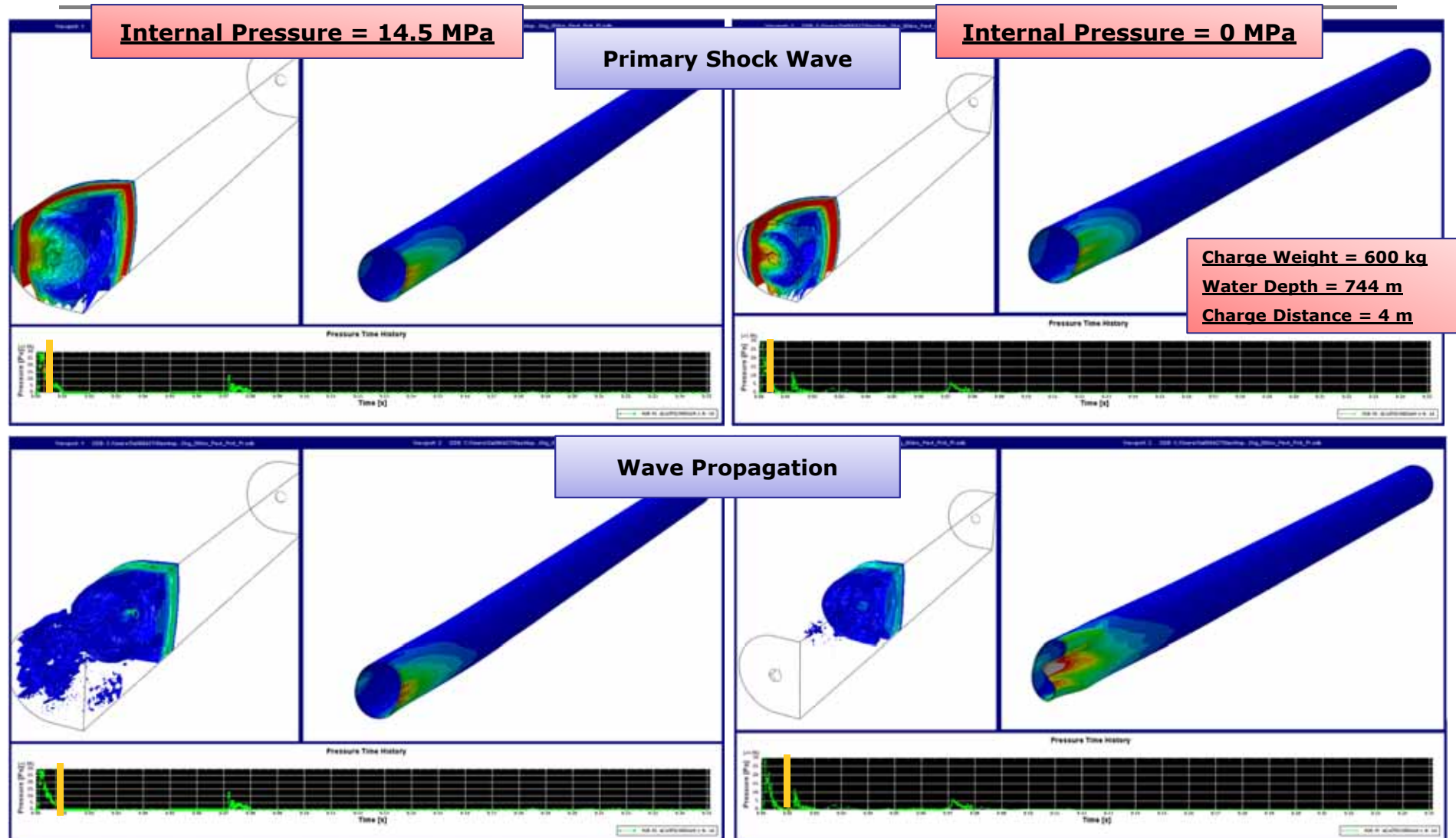
FEM STUDY RESULTS – UNDERWATER EXPLOSION



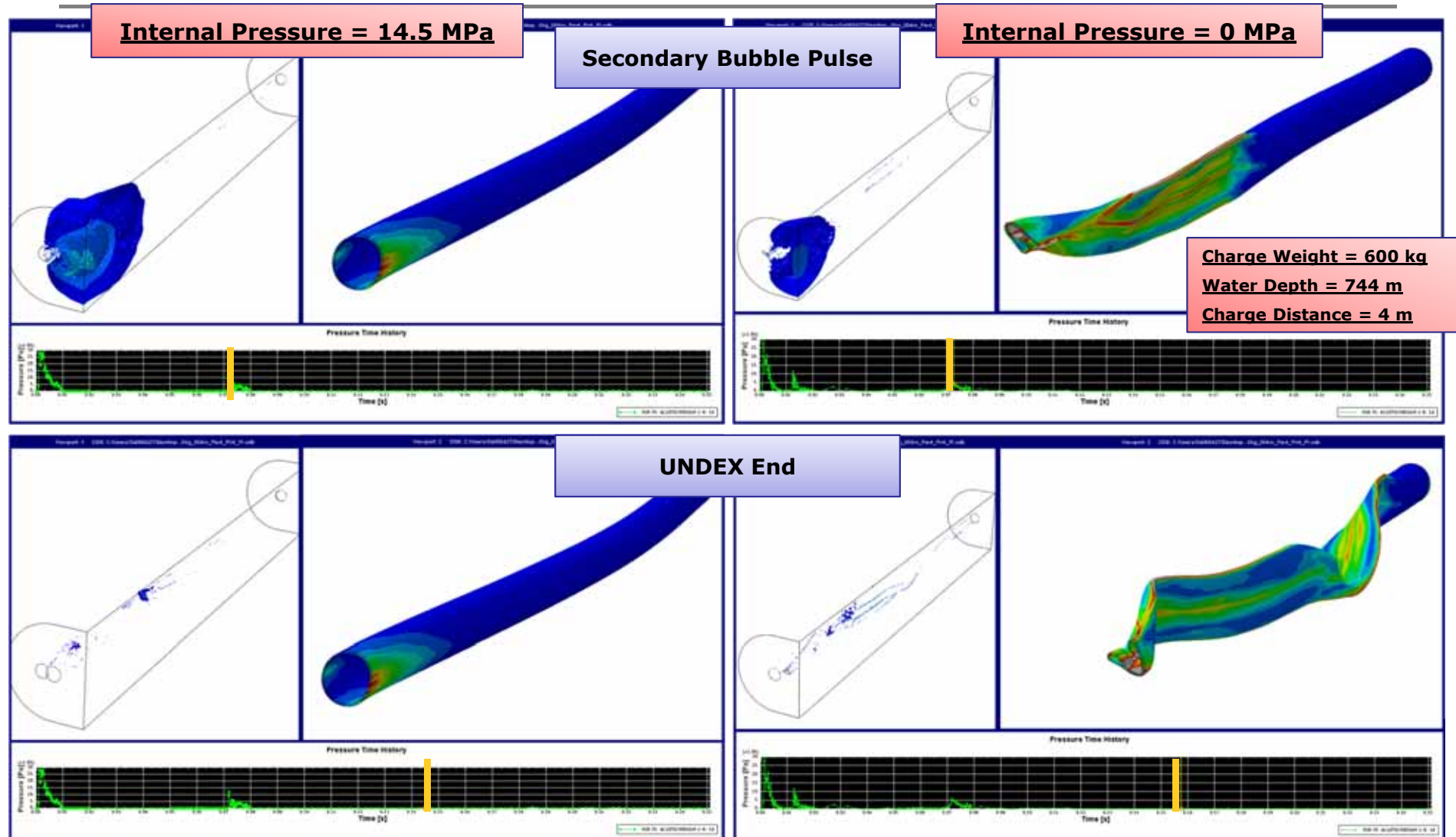
FEM STUDY RESULTS – UNDERWATER EXPLOSION



FEM STUDY RESULTS – PIPELINE DEFORMATION

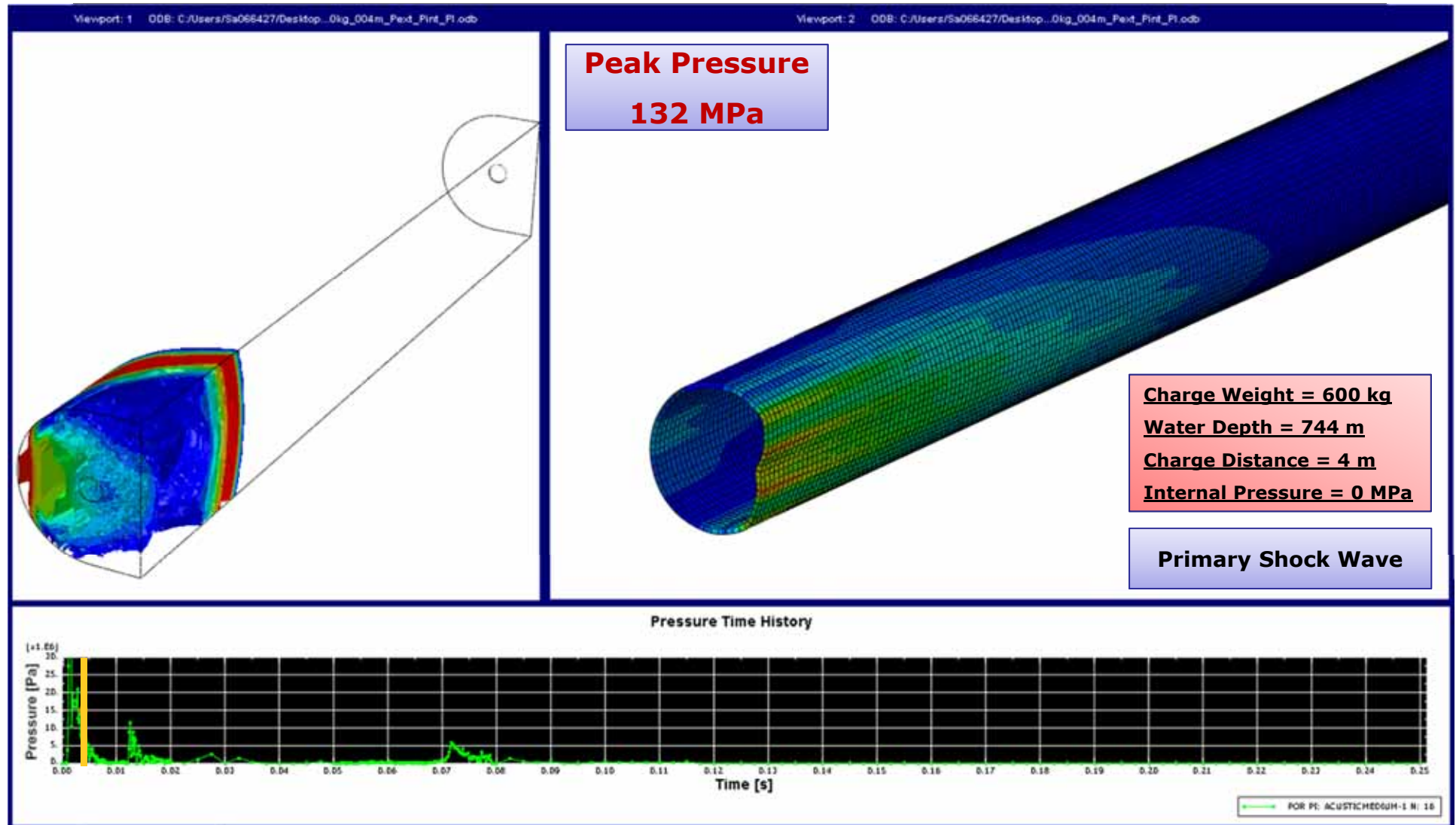


FEM STUDY RESULTS – PIPELINE DEFORMATION



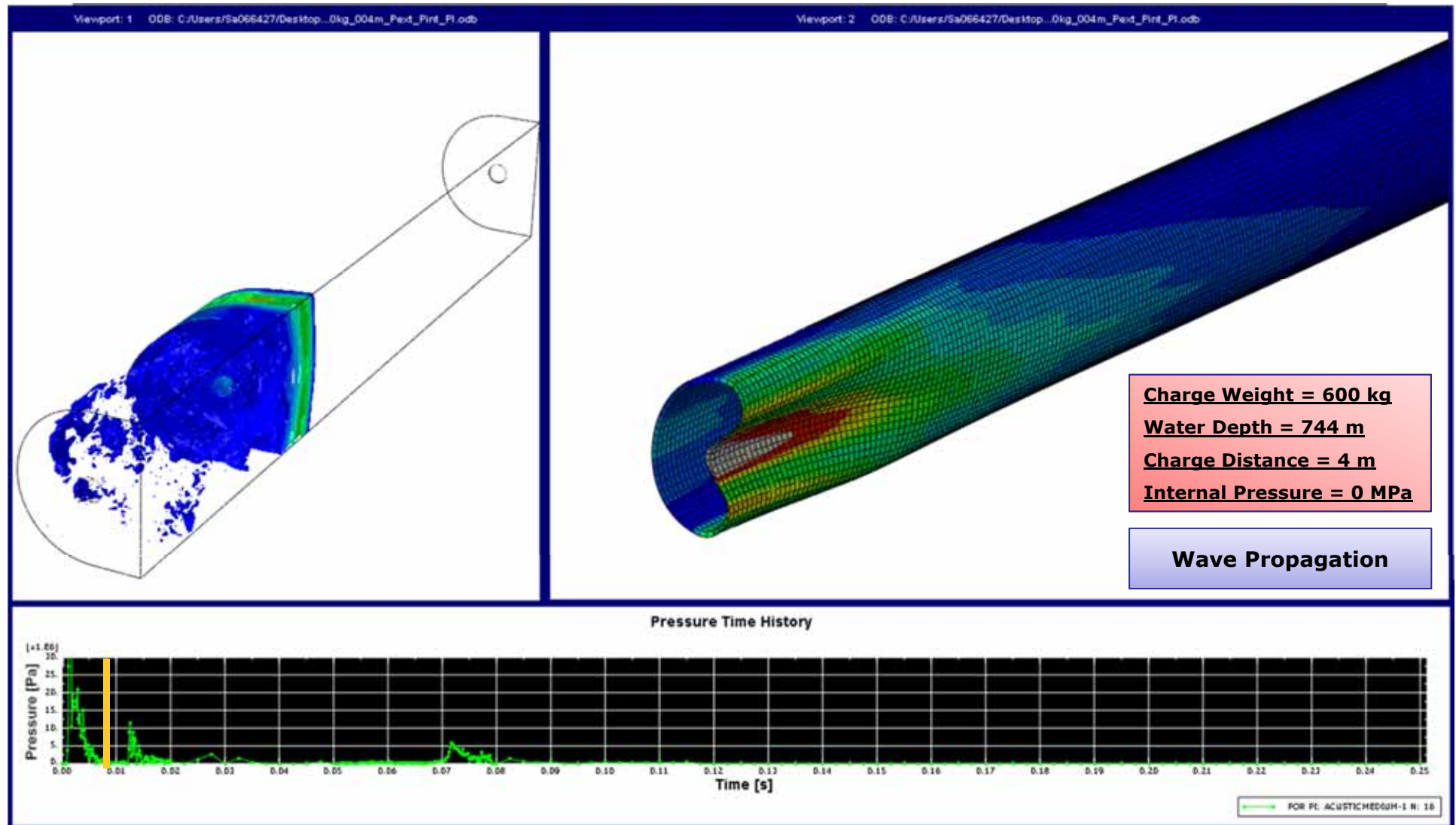
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



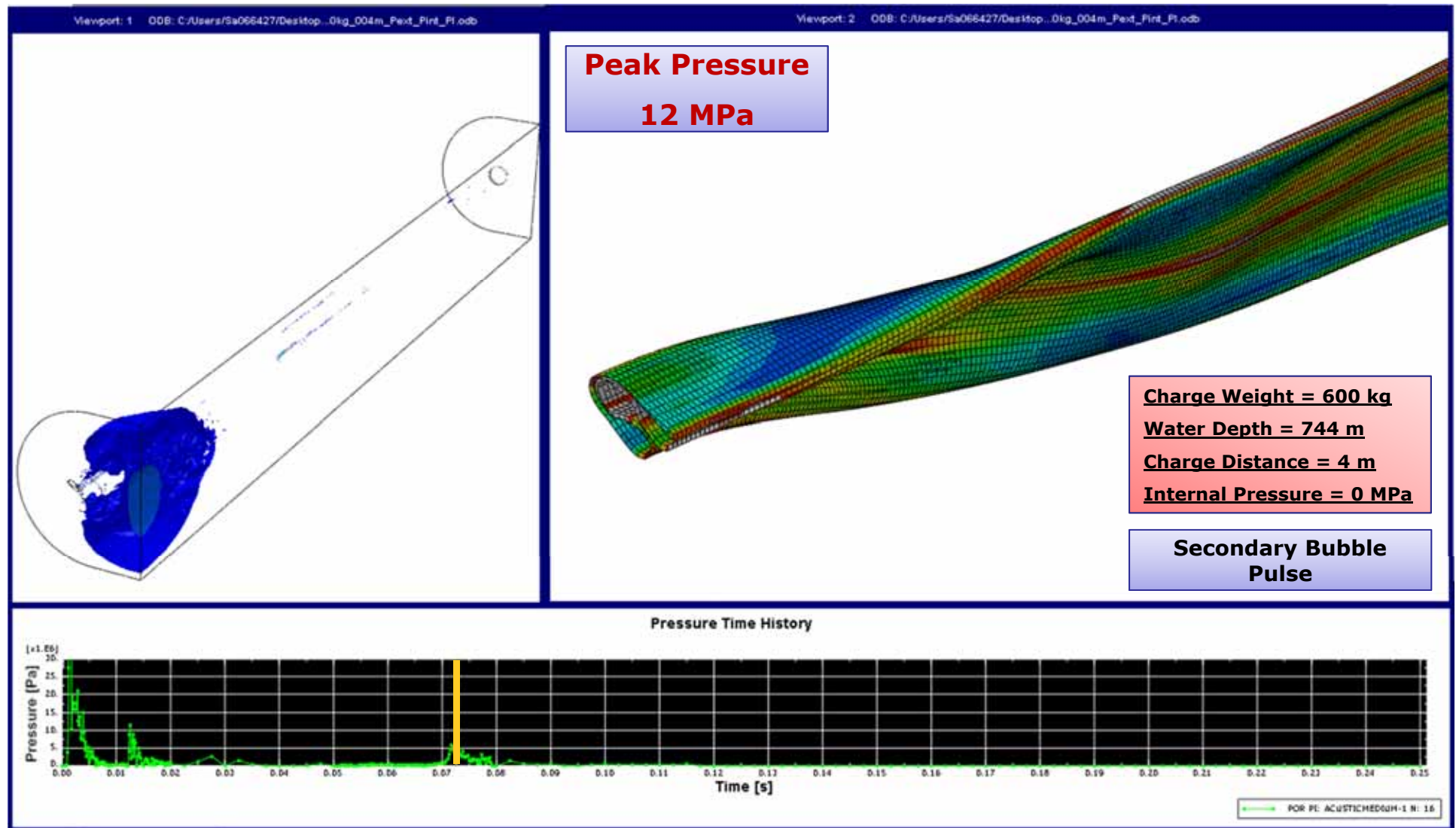
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



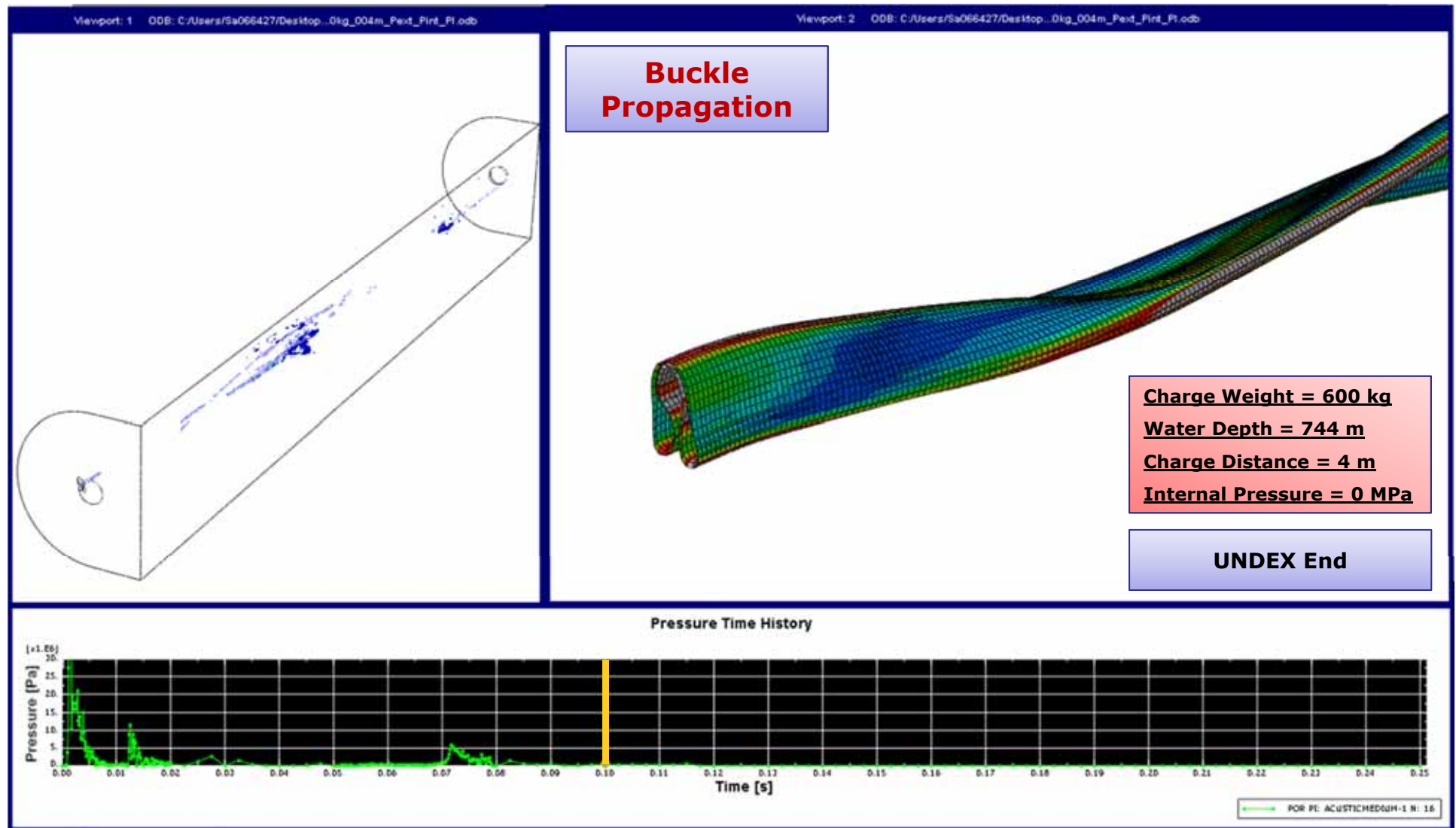
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



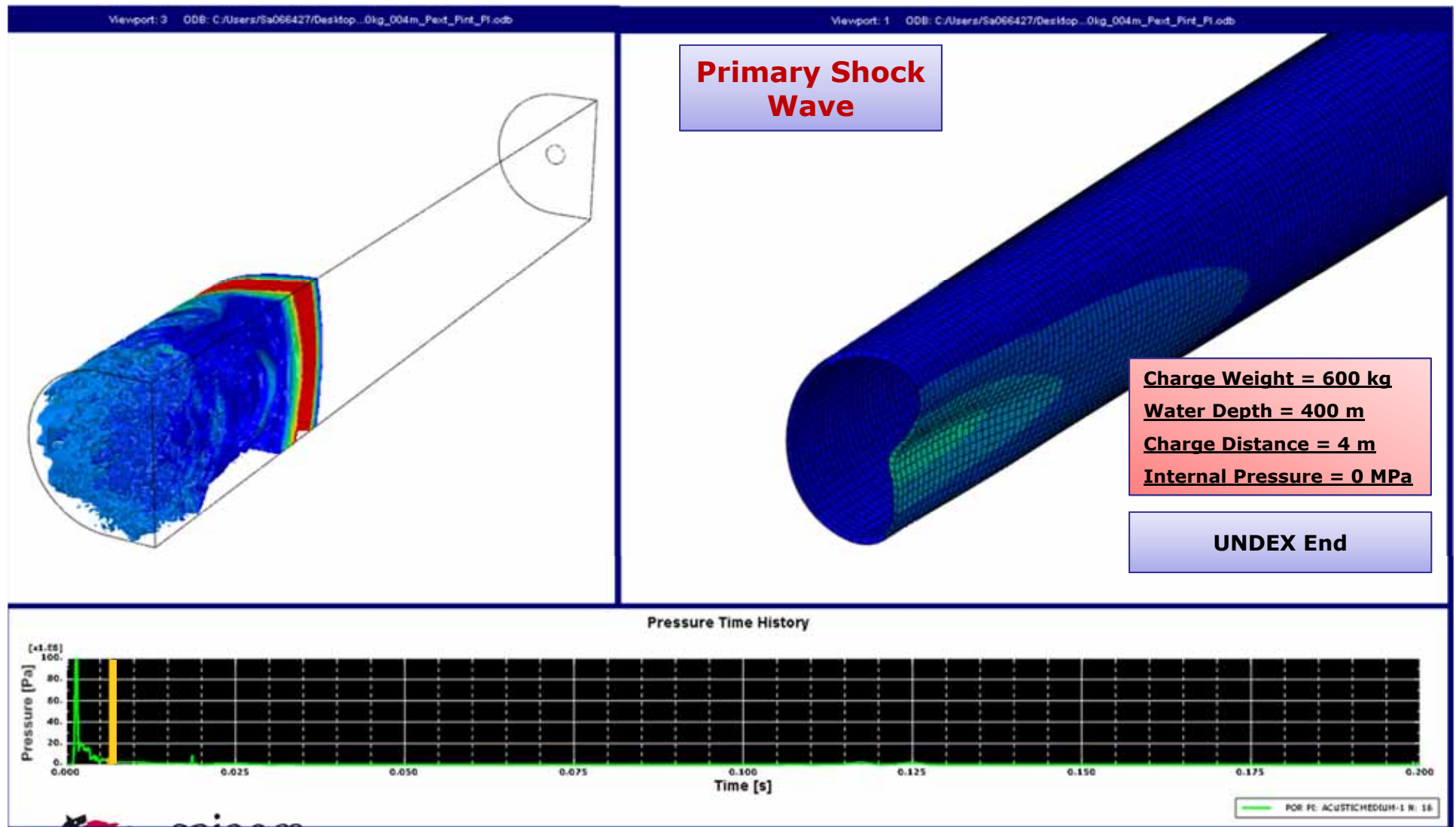
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



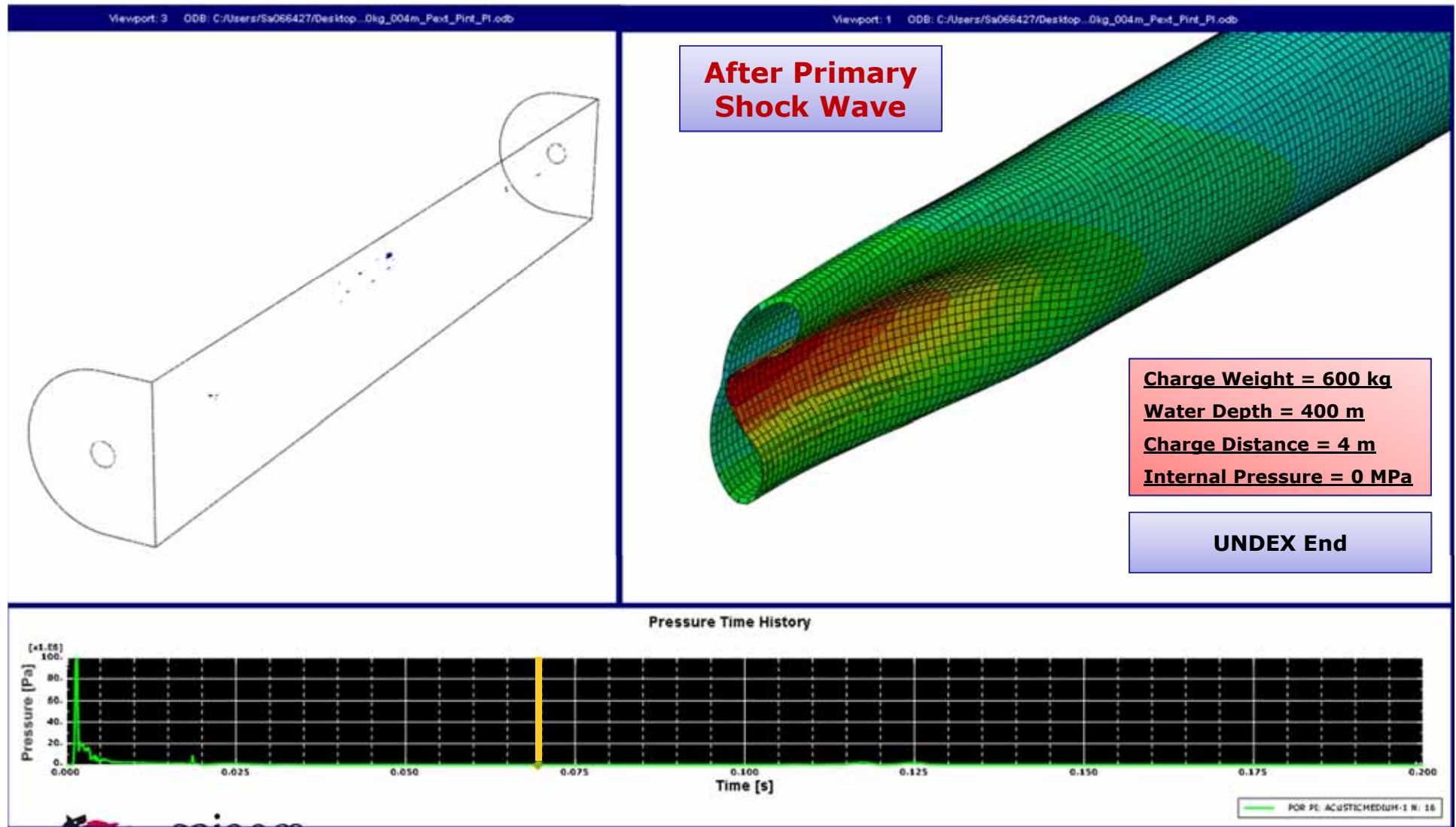
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



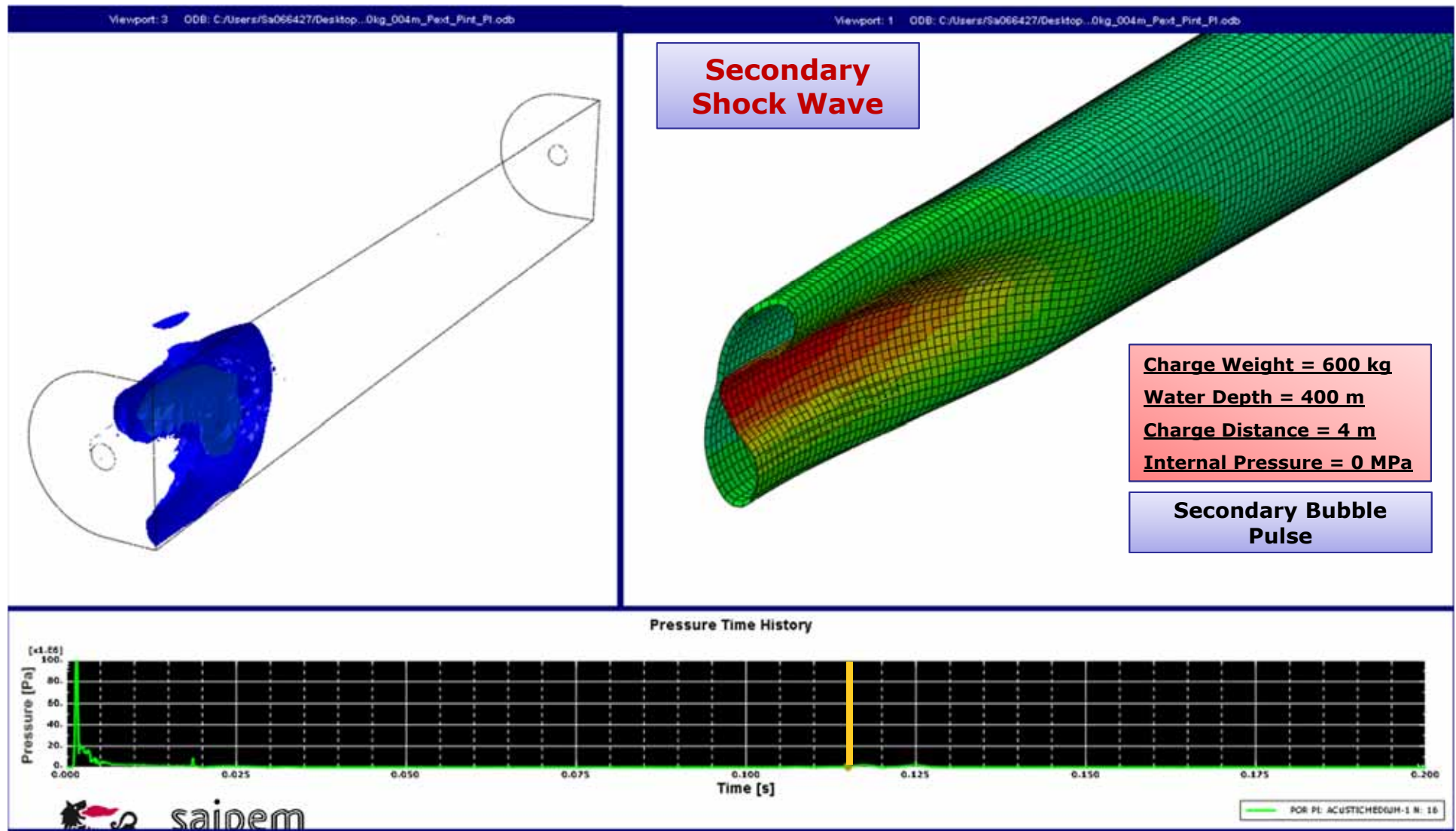
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



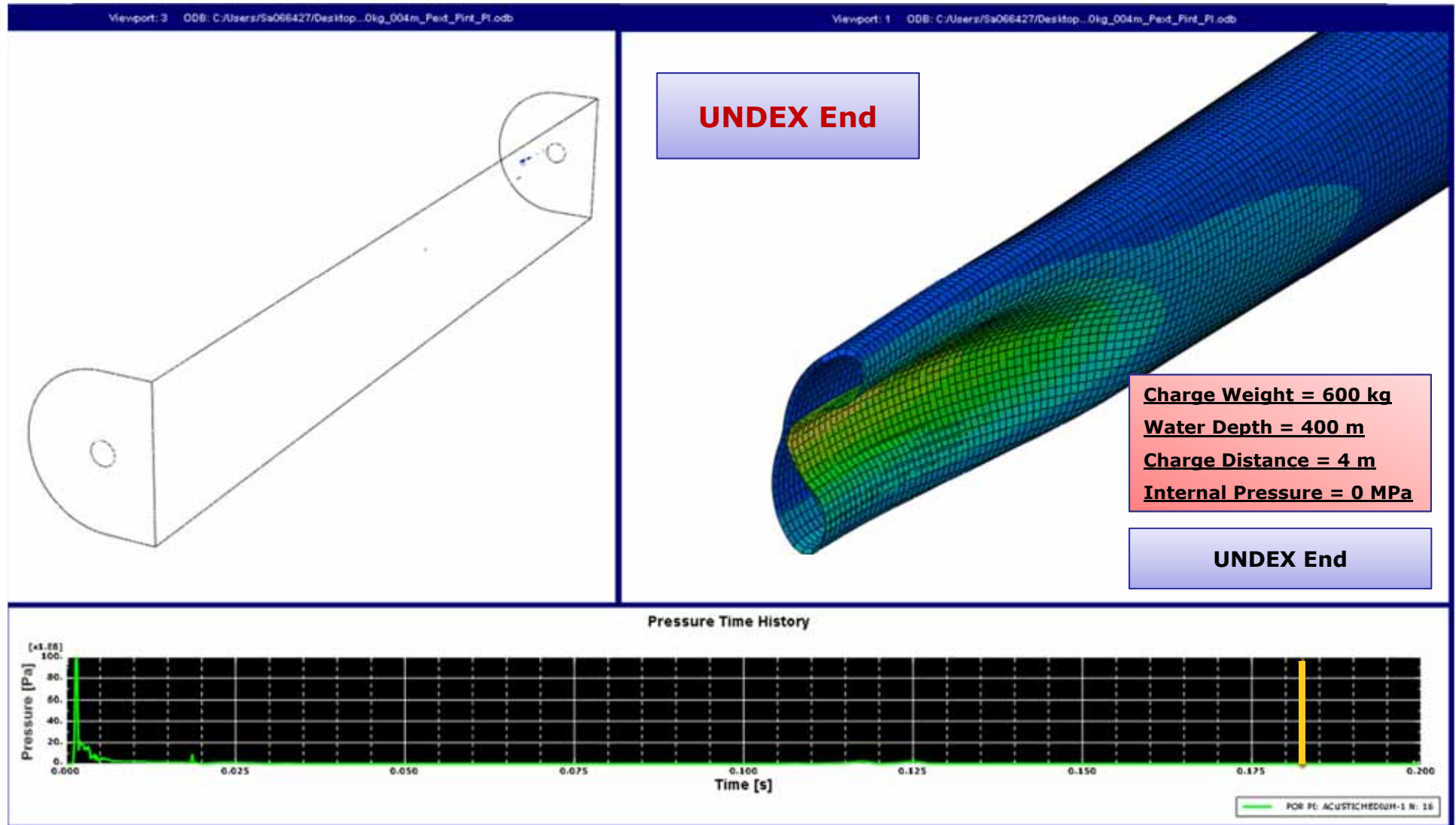
FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure

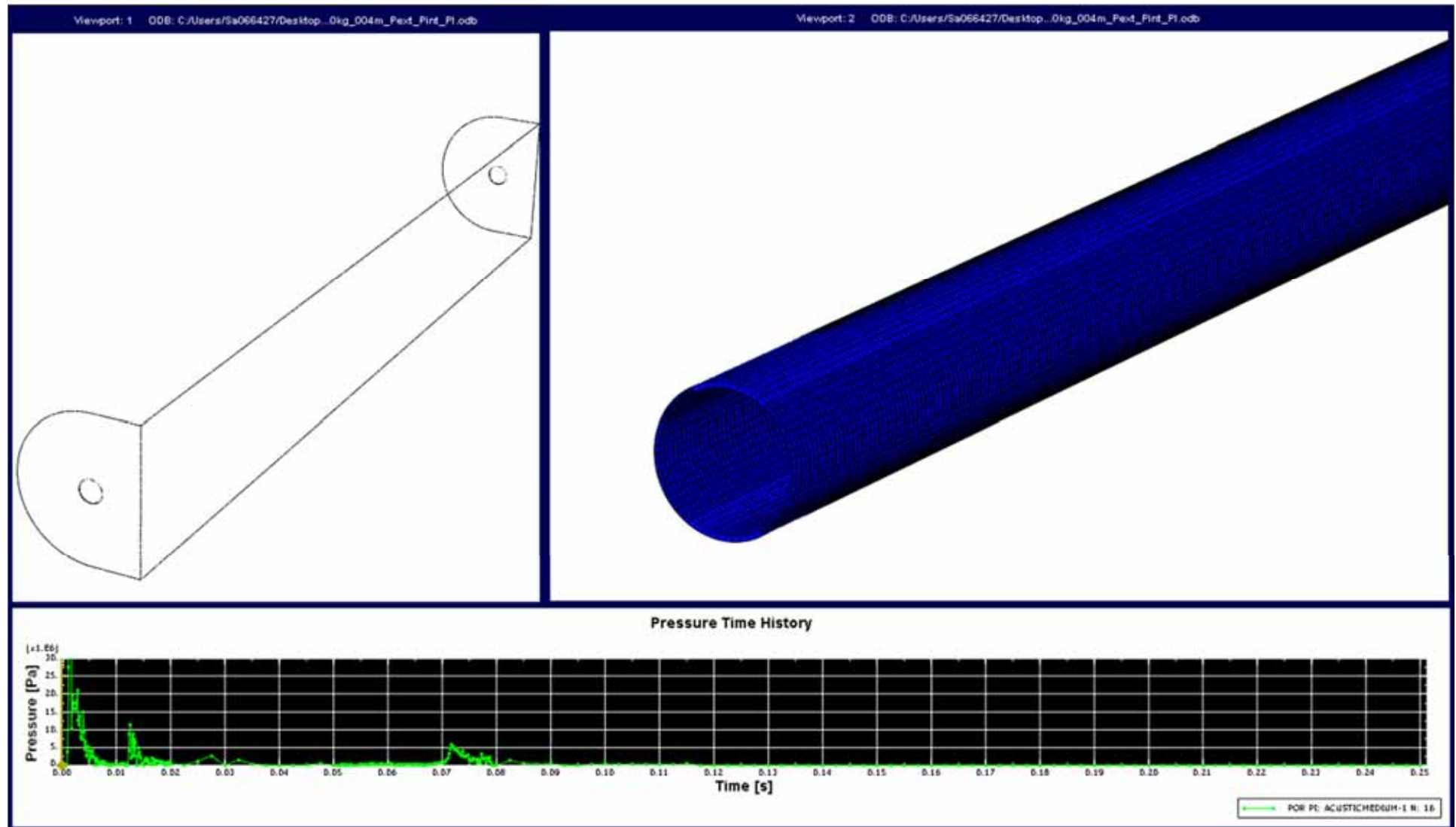


FEM STUDY RESULTS – PIPELINE DEFORMATION

No Internal Pressure



FEM STUDY RESULTS – PIPELINE DEFORMATION



UNDEX – FEM STUDY RESULTS – SUMMARY

Charge Weight = 300 kg Internal Pressure = 0 barg														
Charge Distance (m)	Ovalisat. (%)		Dent (mm)		Dent / Diameter (%)		Longit. Strain (%)		Hoop Strain (%)		Eq. Plastic Strain (%)	Von Mises Stress (MPa)		Max Displ. (m)
	Max	Res	Max	Res	Max	Res	Max	Res	Max	Res	Max	Max	Res	Max
4	11.2	7.7	73	56	8.0	6.2	0.8	0.4	6.3	6.1	7.2	575	411	0.25
5	7.2	4.4	42	29	4.6	3.2	0.6	0.3	3.3	3.1	3.5	544	266	0.18
7.5	4.9	3.2	21	13	2.3	1.4	0.3	0.1	0.9	0.8	0.8	481	178	0.06
10	3.2	2.3	11	6	1.2	0.7	0.1	0.0	0.4	0.3	0.2	441	148	0.04
20	1.5	1.2	2	1	0.3	0.1	0.1	0.0	0.2	0.1	0.0	330	121	0.01
40	1.2	1.0	1	0	0.1	0.0	0.1	0.0	0.1	0.0	0.0	208	114	0.00
130	1.1	1.0	0	0	0.0	0.0	0.0	0.0	0.1	0.0	0.0	126	104	0.00
Charge Weight = 300 kg Internal Pressure = 145 barg														
Charge Distance (m)	Ovalisat. (%)		Dent (mm)		Dent / Diameter (%)		Longit. Strain (%)		Hoop Strain (%)		Eq. Plastic Strain (%)	Von Mises Stress (MPa)		Max Displ. (m)
	Max	Res	Max	Res	Max	Res	Max	Res	Max	Res	Max	Max	Res	Max
4	6.4	3.4	37	23	4.1	2.5	0.6	0.2	3.6	3.3	4.1	550	302	0.28
5	4.5	2.7	21	12	2.3	1.3	0.3	0.2	1.7	1.4	1.7	514	239	0.16
7.5	2.5	1.7	8	3	0.9	0.4	0.2	0.0	0.4	0.1	0.2	439	144	0.07
10	2.0	1.1	5	1	0.6	0.1	0.2	0.0	0.2	0.0	0.1	385	164	0.05
20	1.5	1.1	2	0	0.3	0.0	0.1	0.0	0.1	0.0	0.0	206	94	0.01
40	1.2	1.0	1	0	0.1	0.0	0.1	0.0	0.1	0.1	0.0	144	100	0.00
130	1.1	1.0	0	0	0.0	0.0	0.0	0.0	0.1	0.0	0.0	108	93	0.00
<div> <div>SLS Oval.</div> <div>SLS Dent</div> <div>ULS</div> <div>96% SMYS</div> </div>														

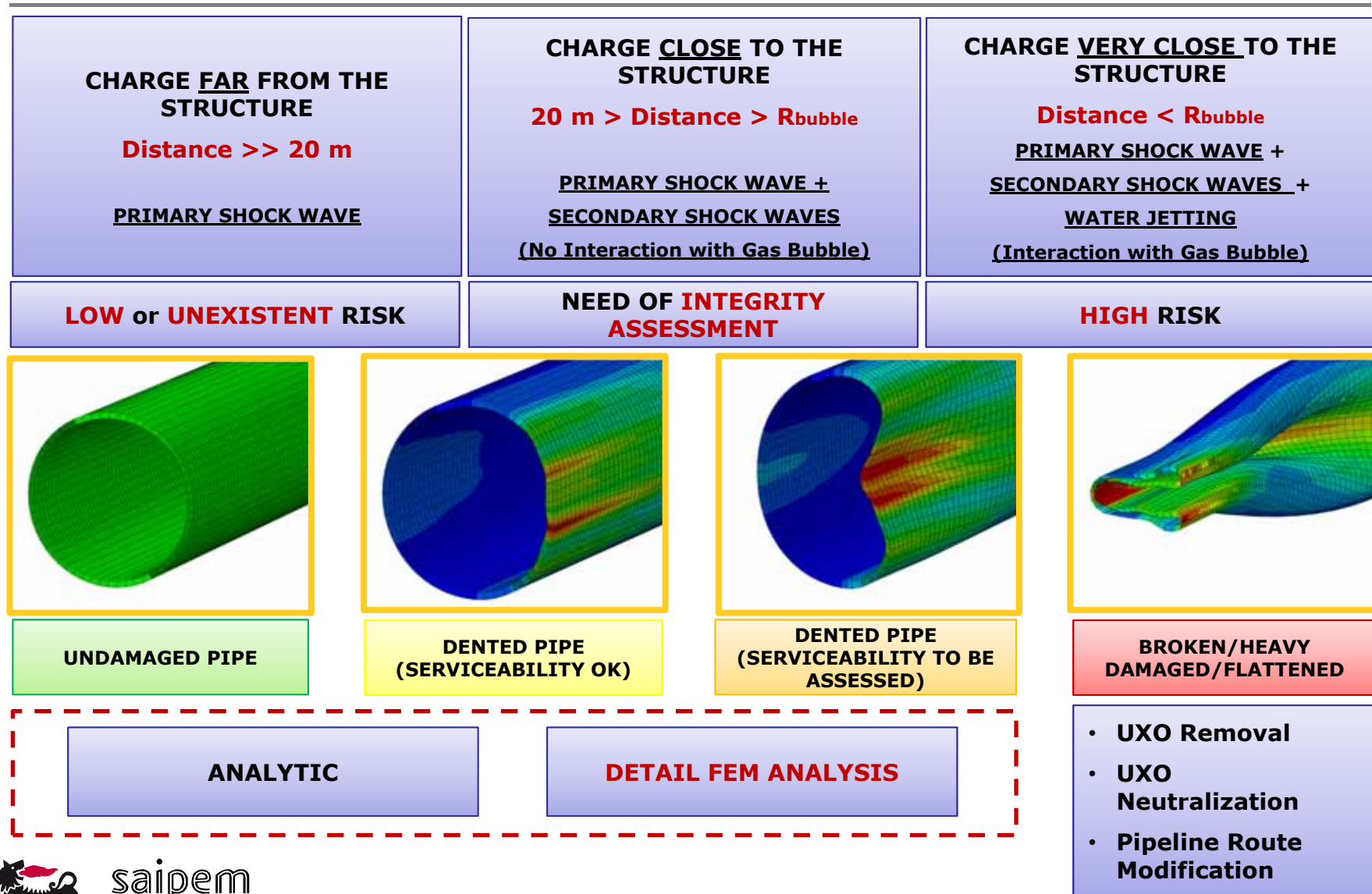


UNDEX – FEM STUDY RESULTS – SUMMARY

Charge Weight = 600 kg Internal Pressure = 0 barg														
Charge Distance (m)	Ovalisat. (%)		Dent (mm)		Dent / Diameter (%)		Longit. Strain (%)		Hoop Strain (%)		Eq. Plastic Strain (%)	Von Mises Stress (MPa)		Max Displ. (m)
	Max	Res	Max	Res	Max	Res	Max	Res	Max	Res	Max	Max	Res	Max
4	NA	NA	NA	NA	NA	NA	NA	NA	NA	NA	NA	NA	NA	NA
5	22.0	20.6	137	130	15.1	14.4	0.8	0.5	10.1	10.0	11.7	597	323	0.54
7.5	6.9	4.2	37	25	4.1	2.7	0.5	0.2	2.4	2.2	2.5	530	199	0.22
10	5.2	3.6	23	15	2.5	1.7	0.4	0.2	1.0	0.8	0.8	485	125	0.08
20	1.7	1.1	4	1	0.4	0.1	0.2	0.0	0.2	0.1	0.3	374	84	0.02
40	1.3	1.1	2	0	0.2	0.0	0.1	0.0	0.1	0.1	0.0	250	116	0.01
130	1.1	1.0	0	0	0.1	0.0	0.0	0.0	0.1	0.0	0.0	137	102	0.00
Charge Weight = 600 kg Internal Pressure = 145 barg														
Charge Distance (m)	Ovalisat. (%)		Dent (mm)		Dent / Diameter (%)		Longit. Strain (%)		Hoop Strain (%)		Eq. Plastic Strain (%)	Von Mises Stress (MPa)		Max Displ. (m)
	Max	Res	Max	Res	Max	Res	Max	Res	Max	Res	Max	Max	Res	Max
4	14.4	5.4	100	56	11.1	6.2	1.1	0.7	10.7	9.8	13.2	600	377	0.77
5	7.4	3.3	46	26	5.1	2.9	0.7	0.3	4.5	4.1	5.2	561	193	0.45
7.5	3.8	2.6	16	9	1.7	1.0	0.3	0.1	1.0	0.7	1.0	489	158	0.18
10	2.5	1.6	8	3	0.9	0.3	0.2	0.1	0.4	0.1	0.2	431	133	0.09
20	1.6	1.1	3	0	0.4	0.0	0.1	0.0	0.1	0.1	0.0	244	100	0.02
40	1.3	1.0	2	0	0.2	0.0	0.1	0.0	0.1	0.0	0.0	163	107	0.01
130	1.1	1.0	1	0	0.1	0.0	0.0	0.0	0.1	0.0	0.0	111	93	0.00
		SLS Oval.				SLS Dent						ULS		96% SMYS



INDEX – FEM STUDY – CONCLUSIONS



FUTURE DEVELOPMENTS

- Refinement of FE Model by considering the **EFFECT OF SEABED** on:
 - Shock Wave Propagation (absorption and reflection of shock waves, crater formation);
 - Effect of actual pipeline embedment.
- Characterization of effect of **EXPLOSIVE CHARGE SHAPE** on explosion behaviour (oriented explosion, shaped charges and Munroe Effect).
- Enhanced modelling of close to pipeline explosion, by considering the interaction between pipeline and **GAS BUBBLE** and the simulation of occurring **WATER JETTING** phenomena. Application of **Coupled Eulerian-Lagrangian ABAQUS** methodology for an enhanced analysis of **FLUID-STRUCTURE INTERACTION**.

THANKS

